HarmonicDrive®
CSF-GH Standard Series

Size
14, 20, 32, 45, 65

5 Sizes

Peak torque
18Nm to 2630Nm

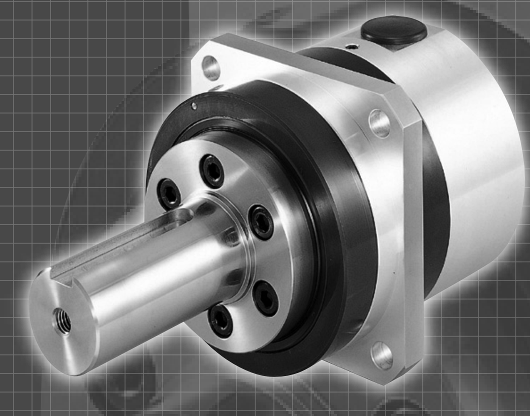
Reduction ratio
50:1 to 160:1

Zero backlash

High Accuracy
Repeatability ±4 to ±10 arc-sec

High Load Capacity Output Bearing
A Cross Roller bearing is integrated with the output flange to provide high moment stiffness, high load capacity and precise positioning accuracy.

Easy mounting to a wide variety of servomotors
Quick Connect® motor adaptation system includes a clamshell style servo coupling and piloted adapter flange.



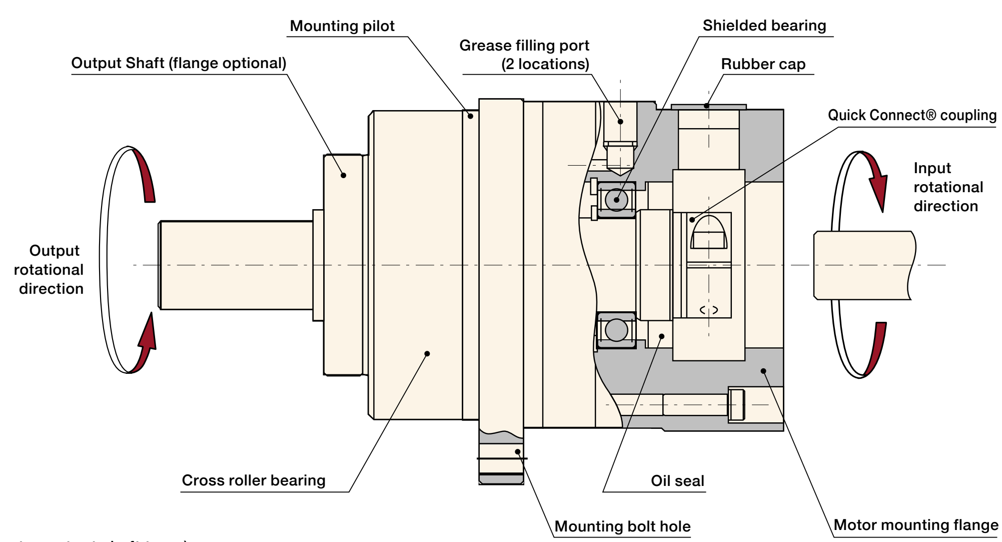
CONTENTS

CSF - 20 - 100 - GH - F0 - Motor Code

Model Name	Size	Reduction Ratio	Model	Output Configuration	Input Configuration
HarmonicDrive® CSF Standard	14	50, 80, 100	GH: Gearhead	F0: Flange output J2: Shaft output without key J6: Shaft output with key and center tapped hole	This code represents the motor mounting configuration. Please contact us for a unique part number based on the motor you are using.
	20	50, 80, 100, 120, 160			
	32				
	45				
	65	80, 100, 120, 160			

Gearhead Construction

Figure 096-1



(The figure indicates output shaft type.)

Rating Table CSF-GH

Table 097-1

Size	Ratio	Rated Torque at 2000 rpm *1	Rated Torque at 3000 rpm *2	Limit for Average Torque *3	Limit for Repeated Peak Torque *4	Limit for Momentary Torque *5	Max. Average Input Speed *6	Max. Input Speed *7	Mass *8	
									Shaft	Flange
		Nm	Nm	Nm	Nm	Nm	rpm	rpm	kg	kg
14	50	5.4	4.7	6.9	18	35	3500	8500	0.62	0.50
	80	7.8	6.8	11	23	47				
	100	7.8	6.8	11	28	54				
20	50	25	22	34	56	98	3500	6500	1.8	1.4
	80	34	30	47	74	127				
	100	40	35	49	82	147				
	120	40	35	49	87	147				
	160	40	35	49	92	147				
32	50	76	66	108	216	382	3500	4800	4.6	3.2
	80	118	103	167	304	568				
	100	137	120	216	333	647				
	120	137	120	216	353	686				
	160	137	120	216	372	686				
45	50	176	154	265	500	950	3000	3800	13	10
	80	313	273	390	706	1270				
	100	353	308	500	755	1570				
	120	402	351	620	823	1760				
	160	402	351	630	882	1910				
65	80	745	651	1040	2110	3720	1900	2800	32	24
	100	951	831	1520	2300	4750				
	120	951	831	1570	2510	4750				
	160	951	831	1570	2630	4750				

*1: Rated torque is based on L10 life of 7,000 hours when input speed is 2000 rpm.
*2: Rated torque is based on L10 life of 7,000 hours when input speed is 3000 rpm, input speed for size 65 is 2800 rpm.
*3: Average load torque calculated based on the application motion profile must not exceed values shown in the table. See p.110.
*4: The limit for torque during start and stop cycles.
*5: The limit for torque during emergency stops or from external shock loads. Always operate below this value.
*6: Max value of average input rotational speed during operation.
*7: Maximum instantaneous input speed.
*8: The mass is for the gearhead only (without input shaft coupling & motor flange). Please contact us for the mass of your specific configuration.

Ratcheting Torque CSF-GH

(Unit: Nm) Table 097-2

Ratio \ Size	14	20	32	45	65
50	88	220	980	2700	–
80	110	350	1400	3900	11000
100	84	260	1000	3100	9400
120	–	240	980	2800	8300
160	–	220	980	2600	8000

Buckling Torque CSF-GH

(Unit: Nm) Table 097-3

Size	14	20	32	45	65
All Ratios	190	560	2200	5800	17000

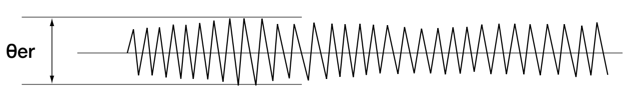
Performance Table CSF-GH

Table 098-1

Size	Flange Type	Ratio	Accuracy*1	Repeatability*2	Starting torque*3	Backdriving torque*4	No-load running torque*5
			arc min	arc sec	Ncm	Nm	Ncm
14	All	50	1.5	±10	8.2	2.9	5.6
		80			6.9	3.9	5.1
		100			6.6	4.7	4.6
20	Type I	50	1.0	±8	13	7.8	11
		80			10	9.6	10
		100			9.6	12	10
		120			9.1	13	9.8
		160			8.6	17	9.6
	Type II & III	50	1.0	±8	20	12	11
		80			17	16	10
		100			16	19	10
		120			16	23	9.8
		160			15	29	9.6
32	Type II	50	1.0	±6	58	35	47
		80			46	44	42
		100			45	54	41
		120			42	61	40
		160			41	79	40
	Type I & III	50	1.0	±6	50	30	47
		80			38	37	42
		100			37	45	41
		120			34	49	40
		160			33	64	40
45	All	50	1.0	±5	123	74	120
		80			95	92	109
		100			89	107	107
		120			85	123	105
		160			79	152	103
65	All	80	1.0	±4	186	179	297
		100			166	200	289
		120			156	226	285
		160			139	268	278

*1: Accuracy values represent the difference between the theoretical angle and the actual angle of output for any given input. The values shown in the table are maximum values.

Figure 098-1

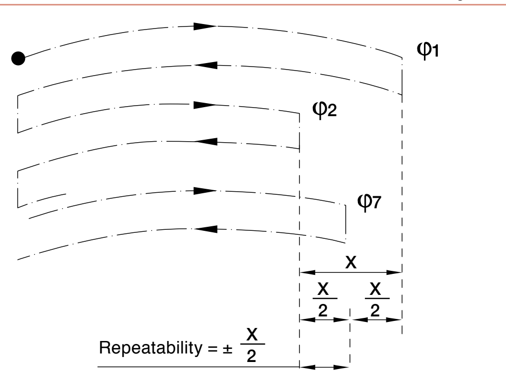


θer : Accuracy
θ1 : Input angle
θ2 : Actual output angle
R : Gear reduction ratio

$$\theta er = \theta_2 - \frac{\theta_1}{R}$$

*2: The repeatability is measured by moving to a given theoretical position seven times, each time approaching from the same direction. The actual position of the output shaft is measured each time and repeatability is calculated as the 1/2 of the maximum difference of the seven data points. Measured values are indicated in angles (arc-sec) prefixed with "±". The values in the table are maximum values.

Figure 098-2



*3: Starting torque is the torque value applied to the input side at which the output first starts to rotate. The values in the table are maximum values.

Table 098-2

Load	No load
Speed reducer surface temperature	25°C

*4: Backdriving torque is the torque value applied to the output side at which the input first starts to rotate. The values in the table are maximum values.

Note: Never rely on these values as a margin in a system that must hold an external load. A brake must be used where back driving is not permissible.

Table 098-3

Load	No load
Speed reducer surface temperature	25°C

*5: No-load running torque is the torque required at the input to operate the gearhead at a given speed under a no-load condition. The values in the table are average values.

Table 098-4

Input speed	2000 rpm
Load	No load
Speed reducer surface temperature	25°C

Torsional Stiffness CSF-GH

Table 099-1

Symbol		Size	14	20	32	45	65
T_1		Nm	2.0	7.0	29	76	235
		kgfm	0.2	0.7	3.0	7.8	24
T_2		Nm	6.9	25	108	275	843
		kgfm	0.7	2.5	11	28	86
Reduction ratio 50	K_1	$\times10^4$Nm/rad	0.34	1.3	5.4	15	—
		kgfm/arc min	0.1	0.38	1.6	4.3	—
	K_2	$\times10^4$Nm/rad	0.47	1.8	7.8	20	—
		kgfm/arc min	0.14	0.52	2.3	6.0	—
	K_3	$\times10^4$Nm/rad	0.57	2.3	9.8	26	—
		kgfm/arc min	0.17	0.67	2.9	7.6	—
	θ_1	$\times10^{-4}$rad	5.8	5.2	5.5	5.2	—
		arc min	2.0	1.8	1.9	1.8	—
	θ_2	$\times10^{-4}$rad	16	15.4	15.7	15.1	—
		arc min	5.6	5.3	5.4	5.2	—
Reduction ratio 80 or more	K_1	$\times10^4$Nm/rad	0.47	1.6	6.7	18	54
		kgfm/arc min	0.14	0.47	2.0	5.4	16
	K_2	$\times10^4$Nm/rad	0.61	2.5	11	29	88
		kgfm/arc min	0.18	0.75	3.2	8.5	26
	K_3	$\times10^4$Nm/rad	0.71	2.9	12	33	98
		kgfm/arc min	0.21	0.85	3.7	9.7	29
	θ_1	$\times10^{-4}$rad	4.1	4.4	4.4	4.1	4.4
		arc min	1.4	1.5	1.5	1.4	1.5
	θ_2	$\times10^{-4}$rad	12	11.3	11.6	11.1	11.3
		arc min	4.2	3.9	4.0	3.8	3.9

* The values in this table are average values. See page 108 for more information about torsional stiffness.

Hysteresis Loss CSF-GH

Reduction ratio 50: Approx. 5.8X10^{-4} rad (2arc min)

Reduction ratio 80 or more: Approx. 2.9X10^{-4} rad (1arc min)

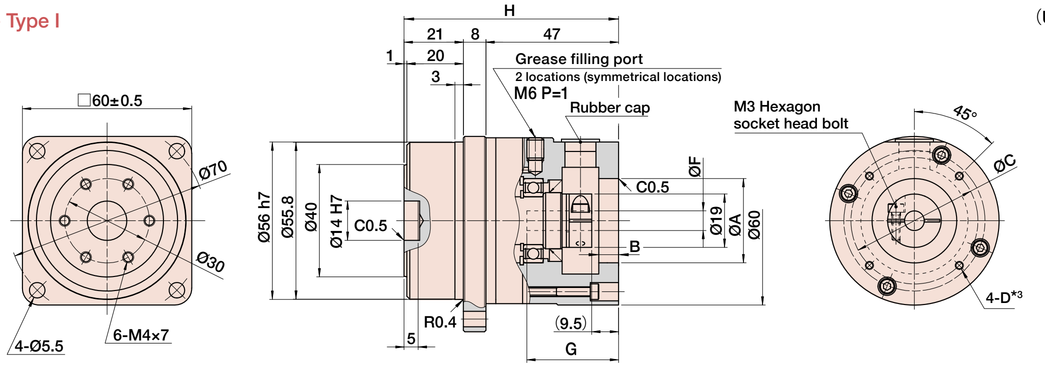
CSF-GH-14 Outline Dimensions

Only primary dimensions are shown in the drawings below. Refer to the confirmation drawing for detailed dimensions.

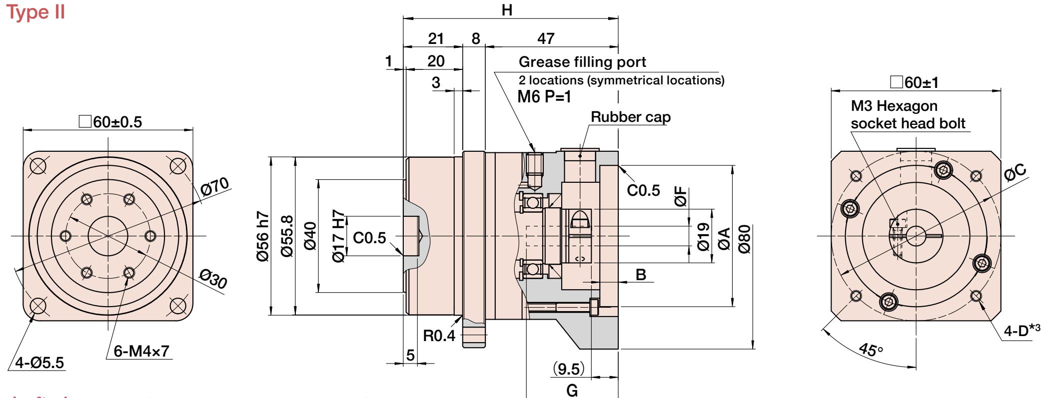
Figure 100-1

(Unit: mm)

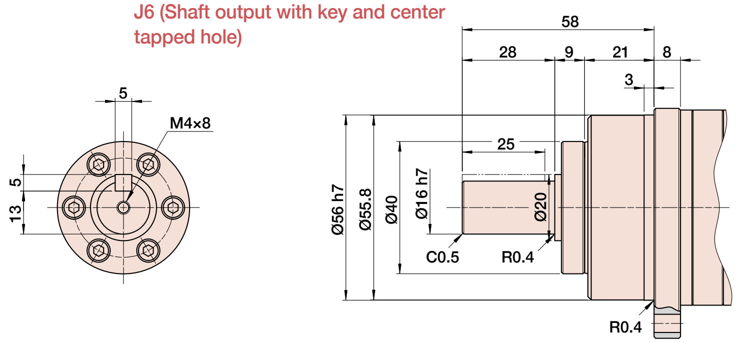
Flange Type I



Flange Type II



Output shaft shape: J2 (Shaft output without key)
J6 (Shaft output with key and center tapped hole)



(Note) The dimension tolerances that are not specified vary depending on the manufacturing method.
Please check the confirmation drawing or contact us for dimension tolerances not shown on the drawing above.

Dimension Table

(Unit: mm) Table 092-1

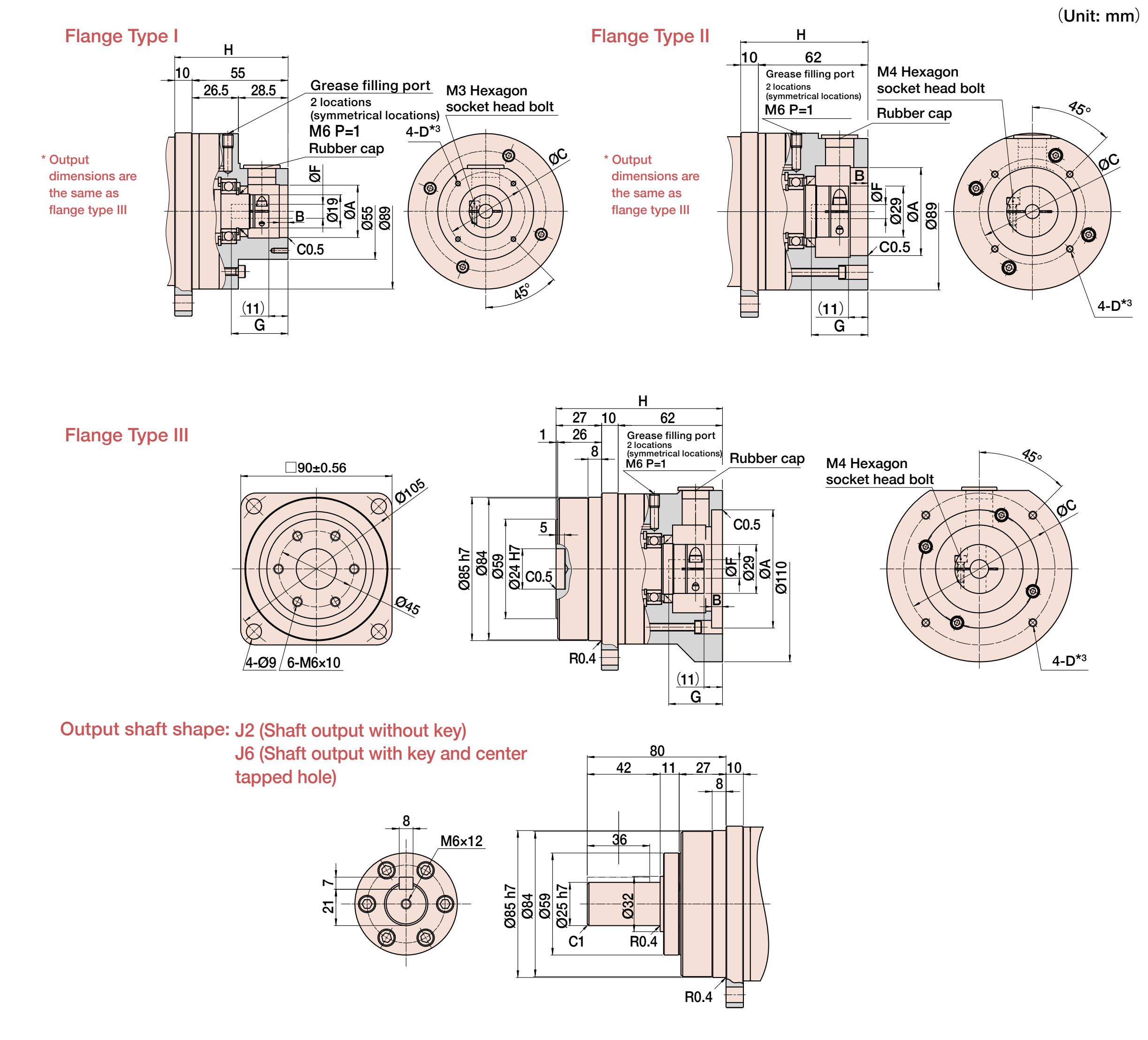
Flange	Coupling	A (H7) *1		B *1	C *1		F (H7) *1		G *1		H *1	Moment of Inertia	Mass (kg) *2	
		Min.	Max.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Typical	($10^{-4}kgm^2$)	Shaft	Flange
Type I	1	30	50	6.5	35	55	6.0	8	20.5	32.5	76	0.07	0.88	0.76
Type II	1	30	55	7	55	75	6.0	8	20.5	32.5	76	0.07	0.90	0.78

Refer to the confirmation drawing for detailed dimensions.
Dimensions of typical products are shown. Please contact us for other mounting options if the configurations shown above are not suitable for your particular motor.
*1 May vary depending on motor interface dimensions.
*2 The mass will vary slightly depending on the ratio and on the inside diameter of the input shaft coupling.
*3 Tapped hole for mounting screw.

CSF-GH-20 Outline Dimensions

Only primary dimensions are shown in the drawings below. Refer to the confirmation drawing for detailed dimensions.

Figure 101-1



(Note) The dimension tolerances that are not specified vary depending on the manufacturing method.
Please check the confirmation drawing or contact us for dimension tolerances not shown on the drawing above.

Dimension Table

(Unit: mm) Table 101-1

Flange	Coupling	A (H7) *1		B *1	C *1		F (H7) *1		G *1		H *1	Moment of Inertia	Mass (kg) *2	
		Min.	Max.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Typical	($10^{-4}kgm^2$)	Shaft	Flange
Type I	1	30	45	5	35	50	7.0	7.8	22	33	92	0.28	2.3	1.9
Type II	2	50	79	10	55	84	8.0	14.6	24	32	99	0.42	2.6	2.2
Type III	2	50	100	10	55	105	8.0	14.6	24	32	99	0.42	2.8	2.4

Refer to the confirmation drawing for detailed dimensions.
Dimensions of typical products are shown. Please contact us for other mounting options if the configurations shown above are not suitable for your particular motor.

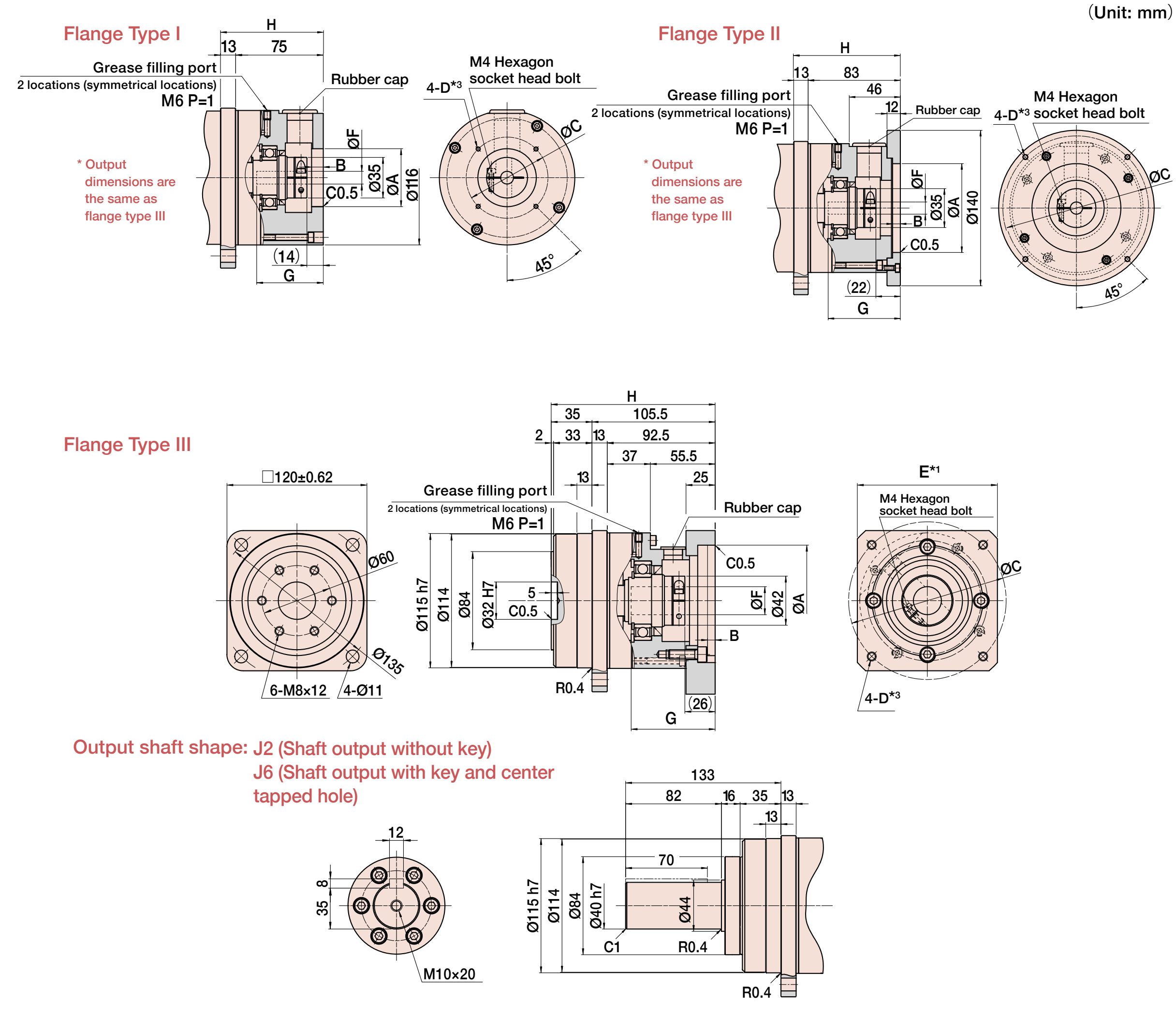
*1 May vary depending on motor interface dimensions.
*2 The mass will vary slightly depending on the ratio and on the inside diameter of the input shaft coupling.
*3 Tapped hole for motor mounting screw.

CSF-GH-32 Outline Dimensions

Only primary dimensions are shown in the drawings below. Refer to the confirmation drawing for detailed dimensions.

Figure 102-1

(Unit: mm)



(Note) The dimension tolerances that are not specified vary depending on the manufacturing method.
Please check the confirmation drawing or contact us for dimension tolerances not shown on the drawing above.

Dimension Table

(Unit: mm) Table 102-1

Flange	Coupling	A (H7) *1		B *1	C *1		F (H7) *1		G *1		H *1	Moment of Inertia	Mass (kg) *2	
		Min.	Max.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Max.	(10^{-4}kgm^2)	Shaft	Flange
Type I	1	50	105	10	55	100	10.8	19.6	27	57	123	2.7	6.4	5.0
	3						8.8	19.6	27	46			6.4	5.0
Type II	2	60	175	5	70	225	16	25.8	39	72	140.5	2.7	7.9	6.5
Type III	1	35	130	7	40	135	10.8	19.6	35	65	131	2.0	6.6	5.2
	3						8.8	19.6	35	54			6.6	5.2

Refer to the confirmation drawing for detailed dimensions.
Dimensions of typical products are shown. Please contact us for other mounting options if the configurations shown above are not suitable for your particular motor.
*1 May vary depending on motor interface dimensions.
*2 The mass will vary slightly depending on the ratio and on the inside diameter of the input shaft coupling.
*3 Tapped hole for motor mounting screw.

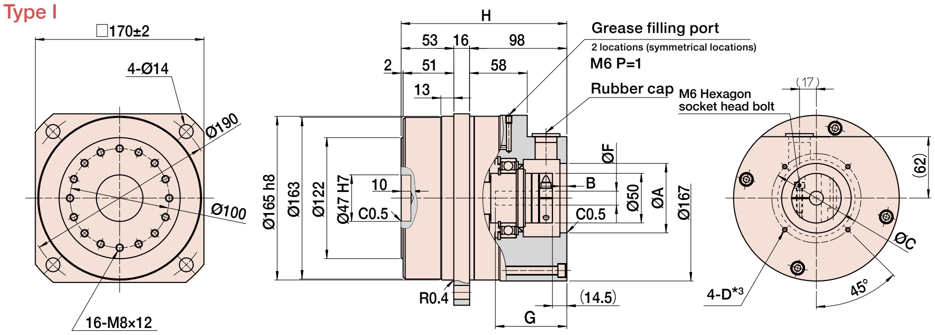
CSF-GH-45 Outline Dimensions

Only primary dimensions are shown in the drawings below. Refer to the confirmation drawing for detailed dimensions.

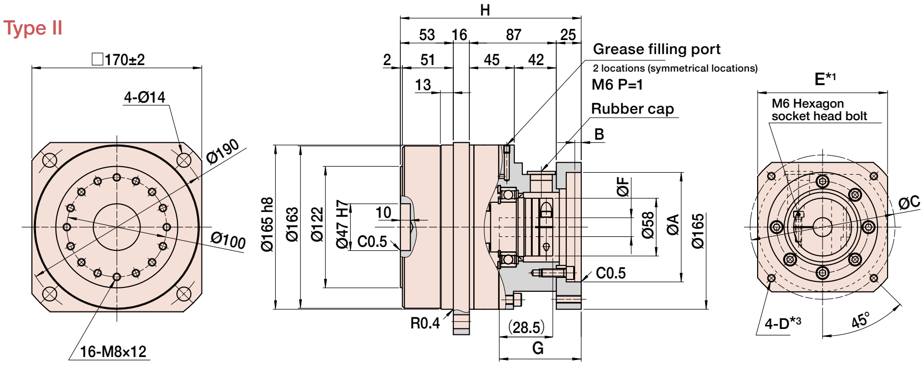
Figure 103-1

(Unit: mm)

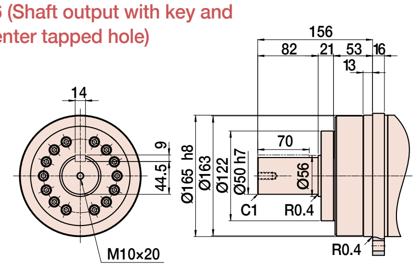
Flange Type I



Flange Type II



Output shaft shape: J2 (Shaft output without key)
J6 (Shaft output with key and center tapped hole)



(Note) The dimension tolerances that are not specified vary depending on the manufacturing method.
Please check the confirmation drawing or contact us for dimension tolerances not shown on the drawing above.

Dimension Table

(Unit: mm) Table 103-1

Flange	Coupling	A (H7) *1		B *1	C *1		F (H7) *1		G *1		H *1	Moment of Inertia	Mass (kg) *2	
		Min.	Max.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Typical	(10⁻⁴kgm²)	Shaft	Flange
Type I	1	70	119	7	80	157	14.0	29.4	30.5	72	167	11	17.3	14.3
Type I	2	70	119	7	80	157	19.0	41	30.5	68	167	11	17.3	14.3
Type II	1	70	175	6.5	80	225	14.0	29.4	44.5	86	181	11	17.7	14.7
Type II	2	70	175	6.5	80	225	19.0	41	44.5	82	181	11	17.7	14.7

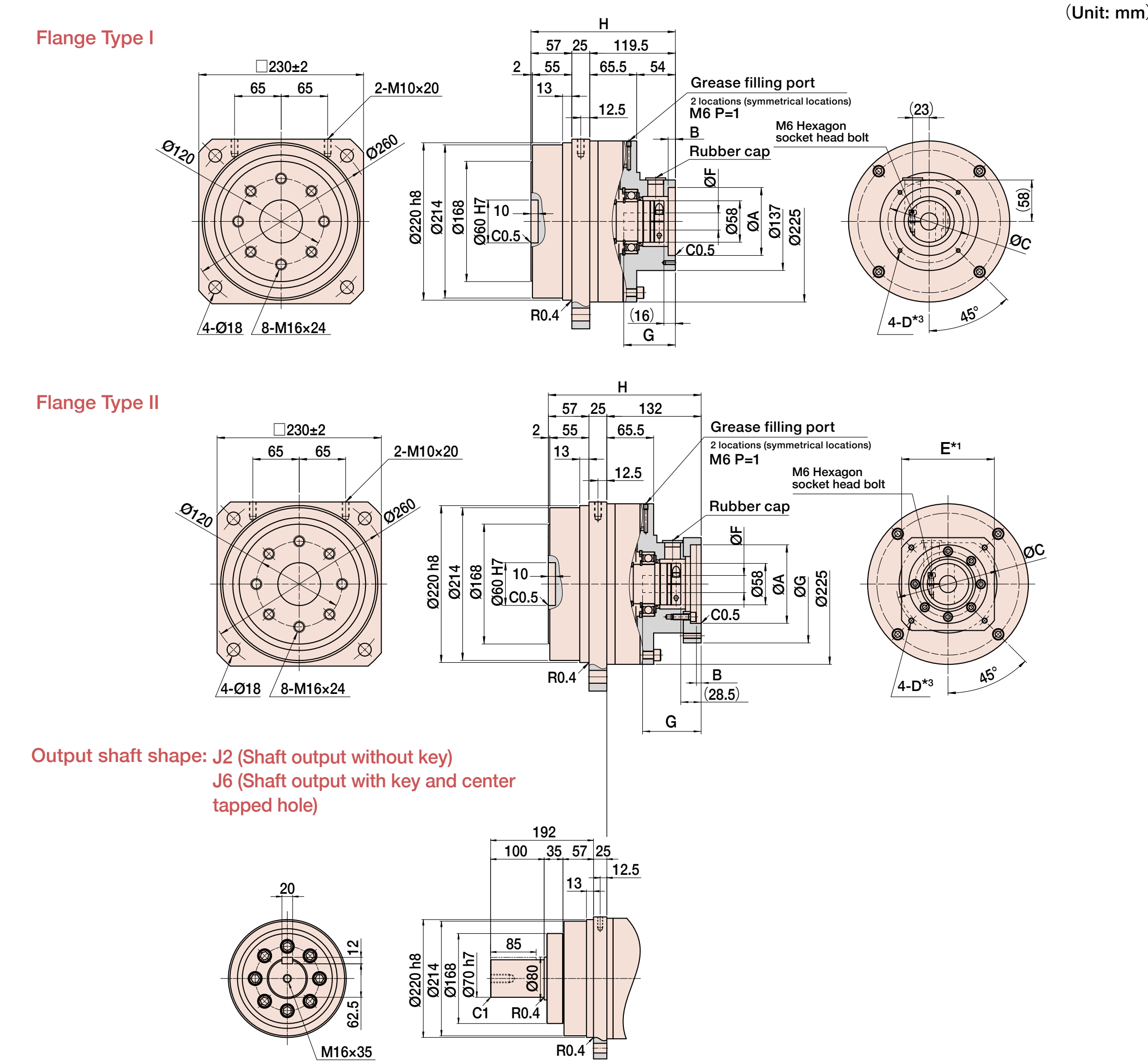
Refer to the confirmation drawing for detailed dimensions.
Dimensions of typical products are shown. Please contact us for other mounting options if the configurations shown above are not suitable for your particular motor.
*1 May vary depending on motor interface dimensions.
*2 The mass will vary slightly depending on the ratio and on the inside diameter of the input shaft coupling.
*3 Tapped hole for motor mounting screw.

CSF-GH-65 Outline Dimensions

Only primary dimensions are shown in the drawings below. Refer to the confirmation drawing for detailed dimensions.

Figure 104-1

(Unit: mm)



(Note) The dimension tolerances that are not specified vary depending on the manufacturing method.
Please check the confirmation drawing or contact us for dimension tolerances not shown on the drawing above.

Dimension Table

(Unit: mm) Table 104-1

Flange	Coupling	A (H7) *1		B *1	C *1		F (H7) *1		G *1		H *1	Moment of Inertia	Mass (kg) *2	
		Min.	Max.	Max.	Min.	Max.	Min.	Max.	Min.	Max.	Max.	(10^{-4}kgm^2)	Shaft	Flange
Type I	1	95	110	10	105	125	19.0	39.3	32.0	72	201.5	51	36.2	27.6
Type II	1	70	215	6.5	80	260	19.0	39.3	44.5	84.5	214	51	38.3	29.7

Refer to the confirmation drawing for detailed dimensions.
Dimensions of typical products are shown. Please contact us for other mounting options if the configurations shown above are not suitable for your particular motor.
*1 May vary depending on motor interface dimensions.
*2 The mass will vary slightly depending on the ratio and on the inside diameter of the input shaft coupling.
*3 Tapped hole for motor mounting screw.

NOTES

Rating Table Definitions

See the corresponding pages of each series for values from the ratings.

■ Rated torque

Rated torque indicates allowable continuous load torque at input speed.

■ Limit for Repeated Peak Torque
(see Graph 106-1)

During acceleration and deceleration the Harmonic Drive® gear experiences a peak torque as a result of the moment of inertia of the output load. The table indicates the limit for repeated peak torque.

■ Limit for Average Torque

In cases where load torque and input speed vary, it is necessary to calculate an average value of load torque. The table indicates the limit for average torque. The average torque calculated must not exceed this limit. (calculation formula: Page 111)

■ Limit for Momentary Torque
(see Graph 106-1)

The gear may be subjected to momentary torques in the event of a collision or emergency stop. The magnitude and frequency of occurrence of such peak torques must be kept to a minimum and they should, under no circumstance, occur during normal operating cycle. The allowable number of occurrences of the momentary torque may be calculated by using the formula on page 111.

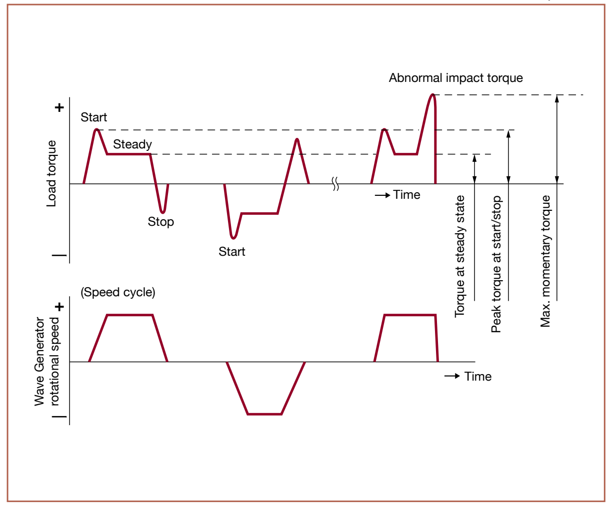
■ Maximum Average Input Speed
Maximum Input Speed

Do not exceed the allowable rating. (calculation formula of the average input speed: Page 111).

■ Inertia

The rating indicates the moment of inertia reflected to the gear input.

Example of load torque pattern Graph 106-1



Life

■ Life of the wave generator

The life of a gear is determined by the life of the wave generator bearing. The life may be calculated by using the input speed and the output load torque.

Table 106-1

Series name	Life	
	CSF-GH	CSG-GH
L_{10}	7,000 hours	10,000 hours
L_{50} (average life)	35,000 hours	50,000 hours

* Life is based on the input speed and output load torque from the ratings.

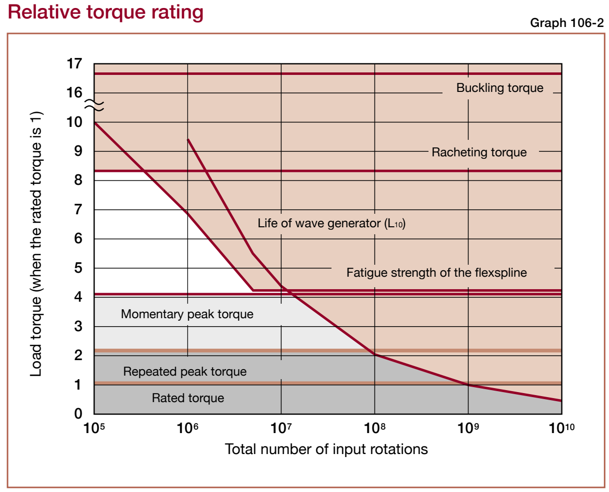
Calculation formula for Rated Lifetime Formula 106-1

$$Lh = Ln \cdot \left(\frac{Tr}{Tav}\right)^3 \cdot \left(\frac{Nr}{Nav}\right)$$

Table 106-2

Ln	Life of L_{10} or L_{50}
Tr	Rated torque
Nr	Rated input speed
Tav	Average load torque on the output side (calculation formula: Page 111)
Nav	Average input speed (calculation formula: Page 111)

Relative torque rating Graph 106-2



* Lubricant life not taken into consideration in the graph described above.
* Use the graph above as reference values.

Torque Limits

■ Strength of flexspline

The Flexspline is subjected to repeated deflections, and its strength determines the torque capacity of the Harmonic Drive® gear. The values given for Rated Torque at Rated Speed and for the allowable Repeated Peak Torque are based on an infinite fatigue life for the Flexspline.
The torque that occurs during a collision must be below the momentary torque (impact torque). The maximum number of occurrences is given by the equation below.

Allowable limit of the bending cycles of the flexspline during rotation of the wave generator while the impact torque is applied: 1.0 x 10^4 (cycles)

The torque that occurs during a collision must be below the momentary torque (impact torque). The maximum number of occurrences is given by the equation below.

Calculation formula — Formula 107-1

$$N = \frac{1.0\times10^4}{2\times\frac{n}{60}\times t}$$

Permissible occurrences	N occurrences
Time that impact torque is applied	t sec
Rotational speed of the wave generator	n rpm
The flexspline bends two times per one revolution of the wave generator.	

Caution — If the number of occurrences is exceeded, the Flexspline may experience a fatigue failure.

■ Buckling torque

When a highly excessive torque (16 to 17 times rated torque) is applied to the output with the input stationary, the flexspline may experience elastic deformation. This is defined as buckling torque.

* See the corresponding pages of each series for buckling torque values.

Warning — When the flexspline buckles, early failure of the HarmonicDrive® gear may occur.

■ Ratcheting torque

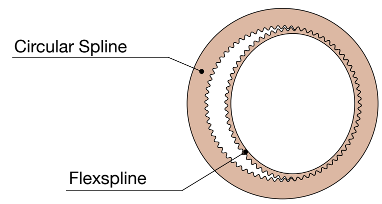
When excessive torque (8 to 9 times rated torque) is applied while the gear is in motion, the teeth between the Circular Spline and Flexspline may not engage properly.
This phenomenon is called ratcheting and the torque at which this occurs is called ratcheting torque. Ratcheting may cause the Flexspline to become non-concentric with the Circular Spline. Operating in this condition may result in shortened life and a Flexspline fatigue failure.

* See the corresponding pages of each series for ratcheting torque values.
* Ratcheting torque is affected by the stiffness of the housing to be used when installing the circular spline. Contact us for details of the ratcheting torque.

Caution — When ratcheting occurs, the teeth may not be correctly engaged and become out of alignment as shown in Figure 099-1. Operating the drive in this condition will cause vibration and damage the flexspline.

Caution — Once ratcheting occurs, the teeth wear excessively and the ratcheting torque may be lowered.

Figure 107-1



"Dedoidal" condition.

Torsional Stiffness

Stiffness and backlash of the drive system greatly affects the performance of the servo system. Please perform a detailed review of these items before designing your equipment and selecting a model number.

■ Stiffness

Fixing the input side (wave generator) and applying torque to the output side (flexspline) generates torsion almost proportional to the torque on the output side. Figure 106-1 shows the torsional angle at the output side when the torque applied on the output side starts from zero, increases up to $+T_0$ and decreases down to $-T_0$. This is called the "Torque – torsion angle diagram," which normally draws a loop of 0 – A – B – A' – B' – A. The slope described in the "Torque – torsion angle diagram" is represented as the spring constant for the stiffness of the HarmonicDrive® gear (unit: Nm/rad).
As shown in Figure 108-2, this "Torque – torsional angle diagram" is divided into 3 regions, and the spring constants in the area are represented by K_1, K_2 and K_3.

K_1 ···· The spring constant when the torque changes from [zero] to [T_1]
K_2 ···· The spring constant when the torque changes from [T_1] to [T_2]
K_3 ···· The spring constant when the torque changes from [T_2] to [T_3]

■ See the corresponding pages of each series for values of the spring constants (K_1, K_2, K_3) and the torque-torsional angles (T_1, T_2, - θ_1, θ_2).

■ Example for calculating the torsion angle

The torsion angle (θ) is calculated here using CSG-32-100-GH as an example.

$T1 = 29$ Nm
$T2 = 108$ Nm
$K1 = 11 \times 10^4$ Nm/rad
$K2 = 12 \times 10^4$ Nm/rad
$K3 = 6.7 \times 10^4$ Nm/rad
$\theta 1 = 4.4 \times 10^{-4}$ rad
$\theta 2 = 11.6 \times 10^{-4}$ rad

When the applied torque is T_1 or less, the torsion angle θ_{L1} is calculated as follows:
When the load torque $T_{L1}=6.0$ Nm

$$\theta_{L1} = T_{L1}/K_1 = 6.0/6.7\times10^4 = 9.0\times10^{-5}\ \text{rad (0.31 arc min)}$$

When the applied torque is between T_1 and T_2, the torsion angle θ_{L2} is calculated as follows:
When the load torque is $T_{L2}=50$ Nm

$$\theta_{L2} = \theta_1 + (T_{L2}-T_1)/K_2 = 4.4\times10^{-4} + (50-29)/11.0\times10^4 = 4.4\times10^{-4} + 1.9\times10^{-4} = 6.3\times10^{-4}\ \text{rad (2.17 arc min)}$$

When the applied torque is greater than T_2, the torsion angle θ_{L3} is calculated as follows:
When the load torque is $T_{L3}=178$ Nm

$$\theta_{L3} = \theta_1 + \theta_2 + (T_{L3}-T_2)/K_3 = 4.4\times10^{-4} + 11.6\times10^{-4} + (178-108)/12.0\times10^4 = 4.4\times10^{-4} + 11.6\times10^{-4} + 5.8\times10^{-4} = 2.18\times10^{-3}\ \text{rad (7.5 arc min)}$$

When a bidirectional load is applied, the total torsion angle will be 2 x θ_{LX} plus hysteresis loss.

* The torsion angle calculation is for the gear component set only and does not include any torsional windup of the output shaft.

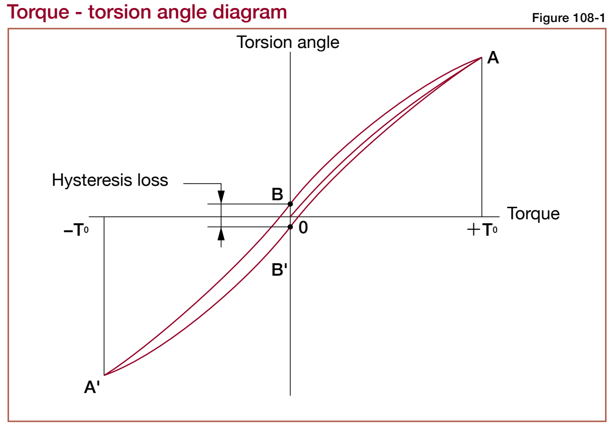
■ Hysteresis loss

As shown in Figure 106-1, when the applied torque is increased to the rated torque and is brought back to [zero], the torsional angle does not return exactly back to the zero point This small difference (B – B') is called hysteresis loss.

■ See the appropriate page for each model series for the hysteresis loss value.

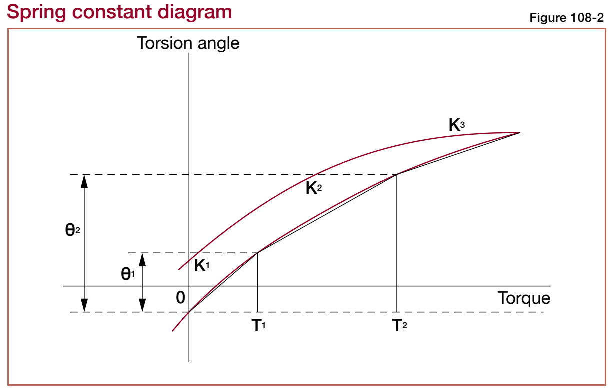
Torque - torsion angle diagram

Figure 108-1



Spring constant diagram

Figure 108-2



■ Backlash

Hysteresis loss is primarily caused by internal friction. It is a very small value and will vary roughly in proportion to the applied load. Because HarmonicDrive® gearheads have zero backlash, the only true backlash is due to the clearance in the Oldham coupling, a self-aligning mechanism used on the wave generator. Since the Oldham coupling is used on the input, the backlash measured at the output is extremely small (arc-seconds) since it is divided by the gear reduction ratio.

Vibration

The primary frequency of the transmission error of the HarmonicDrive® gear may rarely cause a vibration of the load inertia. This can occur when the driving frequency of the servo system including the HarmonicDrive® gear is at, or close to the resonant frequency of the system. Refer to the design guide of each series.

The primary component of the transmission error occurs twice per input revolution of the input. Therefore, the frequency generated by the transmission error is 2x the input frequency (rev / sec).

If the resonant frequency of the entire system, including the HarmonicDrive® gear, is F=15 Hz, then the input speed (N) which would generate that frequency could be calculated with the formula below.

Formula 109-1

$$N = \frac{15}{2} \cdot 60 = 450 \text{ rpm}$$

The resonant frequency is generated at an input speed of 450 rpm.

How to the calculate resonant frequency of the system

Formula 109-2

$$f = \frac{1}{2\pi} \sqrt{\frac{K}{J}}$$

Formula variables

Table 109-1

f	The resonant frequency of the system	Hz	
K	Spring constant	Nm/rad	See pages of each series.
J	Load inertia	kgm^2	

Efficiency

The efficiency will vary depending on the following factors:

- Reduction ratio
- Input speed
- Load torque
- Temperature
- Lubrication condition (Type of lubricant and the quantity)

Product Sizing & Selection

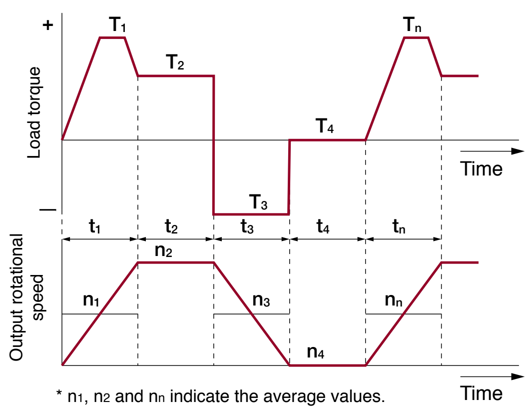
In general, a servo system rarely operates at a continuous load and speed. The input rotational speed, load torque change and comparatively large torque are applied at start and stop. Unexpected impact torque may be applied.
These fluctuating load torques should be converted to the average load torque when selecting a model number.
As an accurate cross roller bearing is built in the direct external load support (output flange), the maximum moment load, life of the cross roller bearing and the static safety coefficient should also be checked.

(Note) If HarmonicDrive® CSG-GH or CSF-GH series is installed vertically with the output shaft facing downward (motor mounted above it) and continuously operated in one direction under the constant load state, lubrication failure may occur. In this case, please contact us for details.

■ Application Motion Profile

Review the application motion profile. Check the specifications shown in the figure below.

Graph 110-1



* n_1, n_2 and n_n indicate the average values.

Obtain the value of each application motion profile.

Load torque	T_n (Nm)
Time	t_n (sec)
Output rotational speed	n_n (rpm)
Normal operation pattern	
Starting (acceleration)	T_1, t_1, n_1
Steady operation (constant velocity)	T_2, t_2, n_2
Stopping (deceleration)	T_3, t_3, n_3
Idle	T_4, t_4, n_4
Maximum rotational speed	
Max. output speed	no ***max***
Max. input rotational speed (Restricted by motors)	ni ***max***
Emergency stop torque	
When impact torque is applied	T_s, t_s, n_s
Required life	L_{10} = L (hours)

■ Flowchart for selecting a size

Please use the flowchart shown below for selecting a size. Operating conditions must not exceed the performance ratings.

Calculate the average load torque applied on the output side from the load torque pattern: T*av* (Nm).

$$T_{av} = \sqrt[3]{\frac{n_1 \cdot t_1 \cdot |T_1|^3 + n_2 \cdot t_2 \cdot |T_2|^3 + \cdots n_n \cdot t_n \cdot |T_n|^3}{n_1 \cdot t_1 + n_2 \cdot t_2 + \cdots n_n \cdot t_n}}$$

↓

Make a preliminary model selection with the following conditions.
T*av* ≦ Limit for average torque torque
(See the ratings of each series).

↓

Calculate the average output speed: no *av* (rpm)

$$no\,av = \frac{n_1 \cdot t_1 + n_2 \cdot t_2 + \cdots n_n \cdot t_n}{t_1 + t_2 + \cdots t_n}$$

Obtain the reduction ratio (R). A limit is placed on "ni *max*" by motors.

$$\frac{ni\,max}{no\,max} \geqq R$$

Calculate the average input rotational speed from the average output rotational speed (no *av*) and the reduction ratio (R): ni *av* (rpm)

$$ni\,av = no\,av \cdot R$$

Calculate the maximum input rotational speed from the max. output rotational speed (no *max*) and the reduction ratio (R): ni *max* (rpm)

$$ni\,max = no\,max \cdot R$$

↓

Check whether the preliminary model number satisfies the following condition from the ratings.
Ni *av* ≦ Limit for average speed (rpm)
Ni *max* ≦ Limit for maximum speed (rpm)
→ NG

OK ↓

Check whether T_1 and T_3 are equal to or less than the repeated peak torque specification.
→ NG

OK ↓

Check whether T_s is equal to or less than the the momentary torque specification.
→ NG

OK ↓

Calculate (Ns) the allowable number of rotations during impact torque.

$$N_S = \frac{10^4}{2 \cdot \frac{n_S \cdot R}{60} \cdot t} \quad \cdots\cdots N_S \leqq 1.0 \times 10^4$$

→ NG

OK ↓

Calculate the lifetime.

$$L_{10} = 7{,}000 \cdot \left(\frac{Tr}{Tav}\right)^3 \cdot \left(\frac{nr}{ni\,av}\right) \text{ (hours)}$$

↓

Check whether the calculated lifetime is equal to or more than the life of the wave generator (see Page 106).
→ NG

OK ↓

The model number is confirmed.

NG → Review the operation conditions and model number

Example of model number selection

Load torque	T_n (Nm)
Time	t_n (sec)
Output rotational speed	n_n (rpm)

Normal operation pattern

Starting (acceleration)	T_1 = 400 Nm, t_1 = 0.3 sec, n_1 = 7 rpm
Steady operation (constant velocity)	T_2 = 320 Nm, t_2 = 3 sec, n_2 = 14 rpm
Stopping (deceleration)	T_3 = 200 Nm, t_3 = 0.4 sec, n_3 = 7 rpm
Dwell Idle	T_4 = 0 Nm, t_4 = 0.2 sec, n_4 = 0 rpm

Maximum rotational speed

Max. output rotational speed	no ***max*** = 14 rpm
Max. input rotational speed (Restricted by motors)	ni ***max*** = 1800 rpm

Emergency stop torque

When impact torque is applied: T_s = 500 Nm, t_s = 0.15 sec, n_s = 14 rpm

Required life

L_{10} = 7000 (hours)

Calculate the average load torque applied on the output side of the Harmonic Drive® gear from the load torque pattern: T***av*** (Nm).

$$T_{av} = \sqrt[3]{\frac{7\,\text{rpm}\cdot 0.3\,\text{sec}\cdot|400\text{Nm}|^3+14\,\text{rpm}\cdot 3\,\text{sec}\cdot|320\text{Nm}|^3+7\,\text{rpm}\cdot 0.4\,\text{sec}\cdot|200\text{Nm}|^3}{7\,\text{rpm}\cdot 0.3\,\text{sec}+14\,\text{rpm}\cdot 3\,\text{sec}+7\,\text{rpm}\cdot 0.4\,\text{sec}}}$$

Make a preliminary model selection with the following conditions. T***av*** = 319 Nm ≦ 620 Nm
(Limit for average torque for model number CSF-45-120-GH: See the ratings on Page 97.)
Thus, **CSF-45-120-GH** is tentatively selected.

Calculate the average output rotational speed: no ***av*** (rpm)

$$n_{o\,av} = \frac{7\,\text{rpm}\cdot 0.3\,\text{sec}+14\,\text{rpm}\cdot 3\,\text{sec}+7\,\text{rpm}\cdot 0.4\,\text{sec}}{0.3\,\text{sec} + 3\,\text{sec} + 0.4\,\text{sec} + 0.2\,\text{sec}} = 12\,\text{rpm}$$

Obtain the reduction ratio (R).

$$\frac{1800\,\text{rpm}}{14\,\text{rpm}} = 128.6 \geqq 120$$

Calculate the average input rotational speed from the average output rotational speed (no ***av***) and the reduction ratio (R): ni ***av*** (rpm)

$$n_{i\,av} = 12\,\text{rpm}\cdot 120 = 1440\,\text{rpm}$$

Calculate the maximum input rotational speed from the maximum output rotational speed (no ***max***) and the reduction ratio (R): ni ***max*** (rpm)

$$n_{i\,max} = 14\,\text{rpm}\cdot 120 = 1680\,\text{rpm}$$

Check whether the preliminary selected model number satisfies the following condition from the ratings.
Ni ***av*** = 1440 rpm ≦ 3000 rpm (Max average input speed of size 45)
Ni ***max*** = 1680 rpm ≦ 3800 rpm (Max input speed of size 45)

NG

OK

Check whether T_1 and T_3 are equal to or less than the repeated peak torque specification.
T_1 = 400 Nm ≦ 823 Nm (Limit of repeated peak torque of size 45)
T_3 = 200 Nm ≦ 823 Nm (Limit of repeated peak torque of size 45)

NG

OK

Check whether Ts is equal to or less than the momentary torque specification.
T_s = 500 Nm ≦ 1760 Nm (Limit for momentary torque of size 45)

NG

OK

Calculate the allowable number (Ns) rotation during impact torque and confirm ≦ 1.0×10⁴

$$N_S = \frac{10^4}{2\cdot\frac{14\,\text{rpm}\cdot 120}{60}\cdot 0.15\,\text{sec}} = 1190 \leqq 1.0\times10^4$$

NG

OK

Calculate the lifetime.

$$L_{10} = 7000\cdot\left(\frac{402\,\text{Nm}}{319\,\text{Nm}}\right)^3\cdot\left(\frac{2000\,\text{rpm}}{1440\,\text{rpm}}\right)\,\text{(hours)}$$

Check whether the calculated life is equal to or more than the life of the wave generator (see Page 106).
L_{10} =19,457 hours ≧ 7000 (life of the wave generator: L_{10})

NG

OK

The selection of model number **CSF-45-120-GH** is confirmed from the above calculations.

Review the operation conditions and model number

HarmonicPlanetary®

HarmonicDrive®

Technical Information

Product Handling

The rated value and performance vary depending on the product series. Be sure to check the usage conditions and refer to the items conforming to the related product.

Efficiency

In general, the efficiency of a speed reducer depends on the reduction ratio, input rotational speed, load torque, temperature and lubrication condition. The efficiency of each series under the following measurement conditions is plotted in the graphs on the next page. The values in the graph are average values.

Measurement condition

Table 134-1

Input rotational speed	HPGP / HPG / HPF / HPN：3000rpm CSG-GH / CSF-GH：Indicated on each efficiency graph.
Ambient temperature	25°C
Lubricant	Use standard lubricant for each model. (See pages 163- 164 for details.)

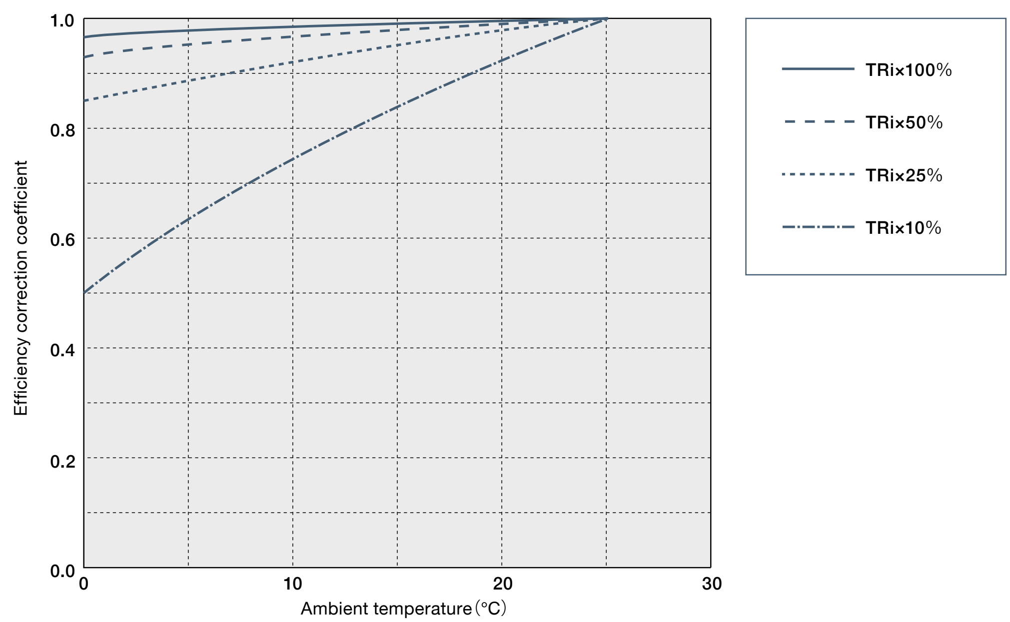
Efficiency compensated for low temperature

Calculate the efficiency at an ambient temperature of 25°C or less by multiplying the efficiency at 25°C by the low-temperature efficiency correction value. Obtain values corresponding to an ambient temperature and to an input torque (TRi*) from the following graphs when calculating the low-temperature efficiency correction value.

HPGP HPG HPF HPN

* TRi is an input torque corresponding to output torque at 25°C.

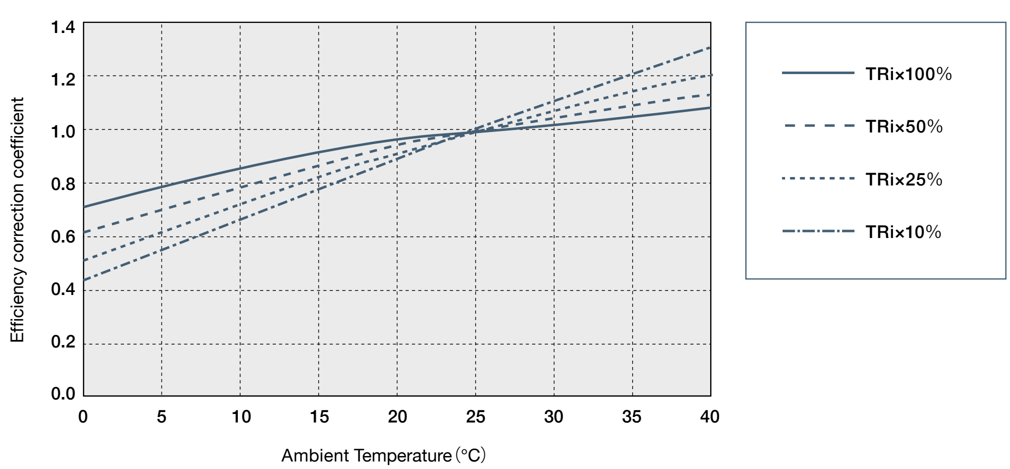
Graph 134-1



CSG-GH CSF-GH

* TRi is an input torque corresponding to output torque at 25°C.

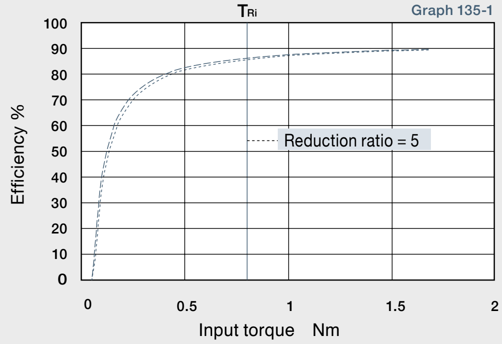
Graph 134-2



Size 11 : Gearhead HPGP

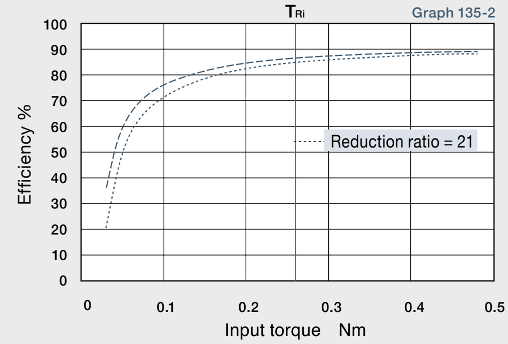
Reduction Ratio = 5

Graph 135-1



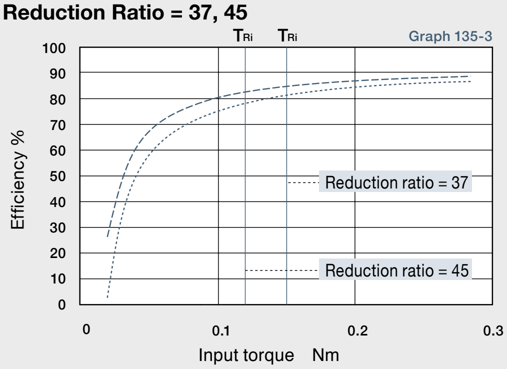
Reduction Ratio = 21

Graph 135-2



Reduction Ratio = 37, 45

Graph 135-3

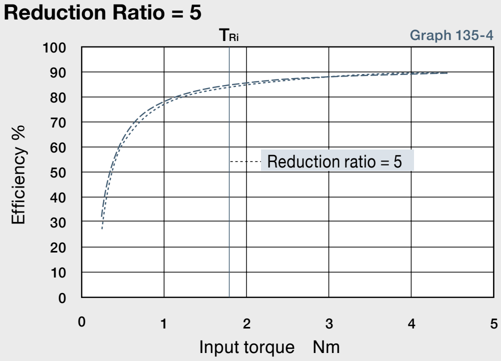


--- Gearhead (standard item) ····· Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

Size 14 : Gearhead HPGP

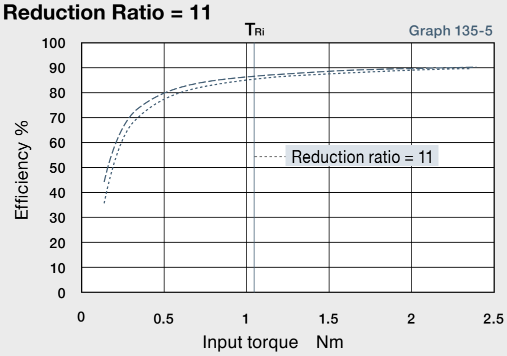
Reduction Ratio = 5

Graph 135-4



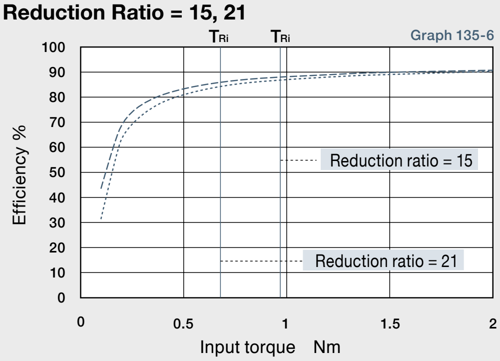
Reduction Ratio = 11

Graph 135-5



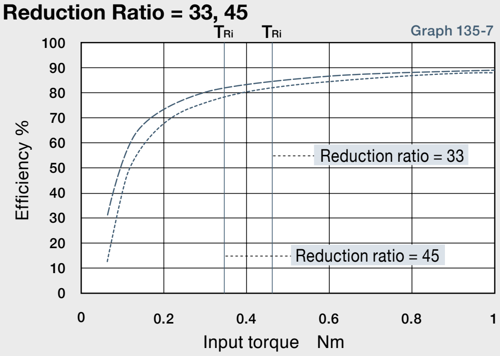
Reduction Ratio = 15, 21

Graph 135-6



Reduction Ratio = 33, 45

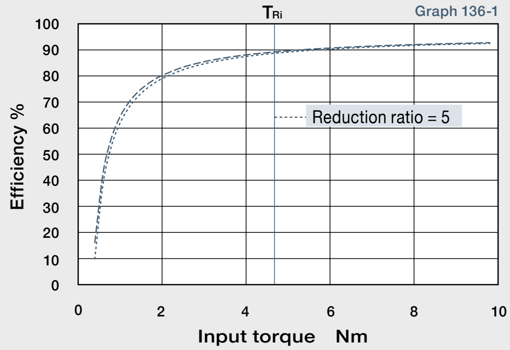
Graph 135-7



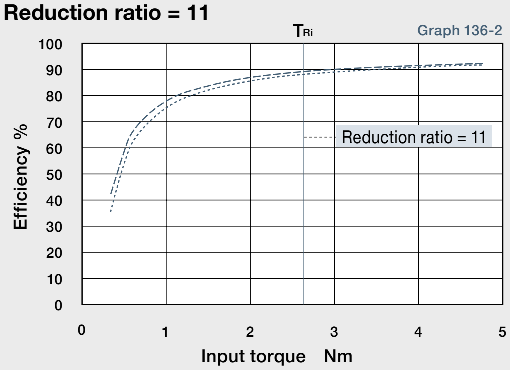
--- Gearhead (standard item) ····· Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

Size 20 : Gearhead HPGP

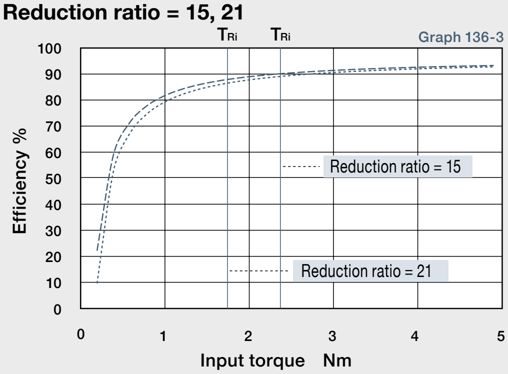
Reduction ratio = 5



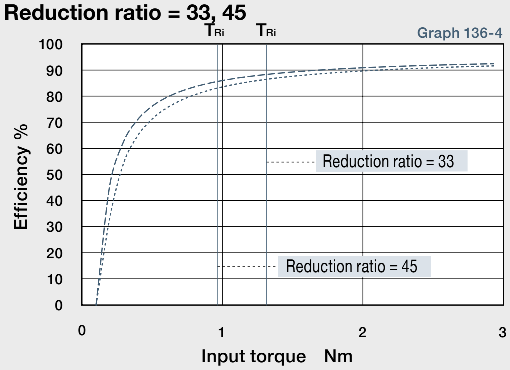
Reduction ratio = 11



Reduction ratio = 15, 21



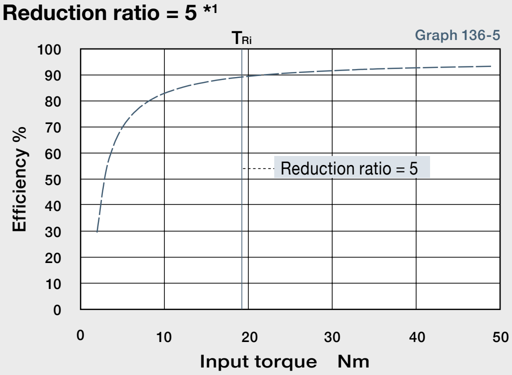
Reduction ratio = 33, 45



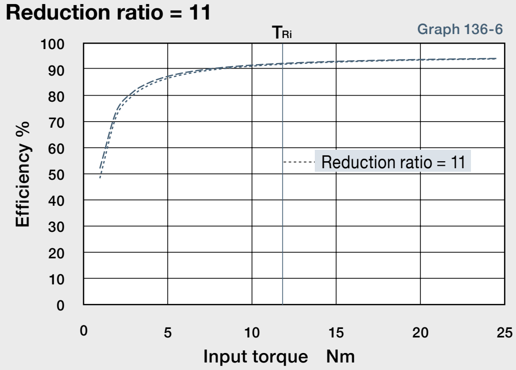
--- Gearhead (standard item) ...... Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

Size 32 : Gearhead HPGP

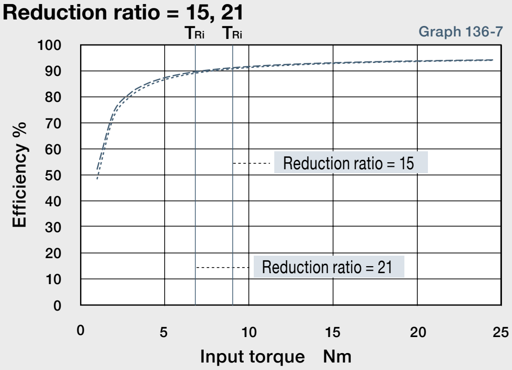
Reduction ratio = 5 *1



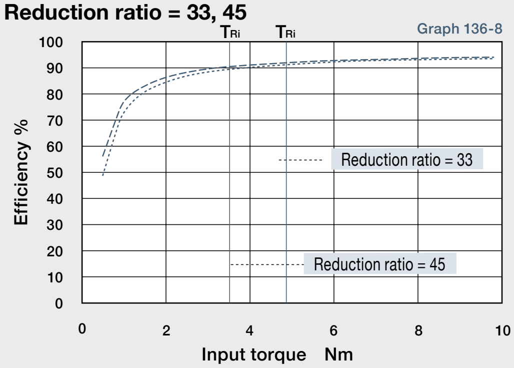
Reduction ratio = 11



Reduction ratio = 15, 21



Reduction ratio = 33, 45

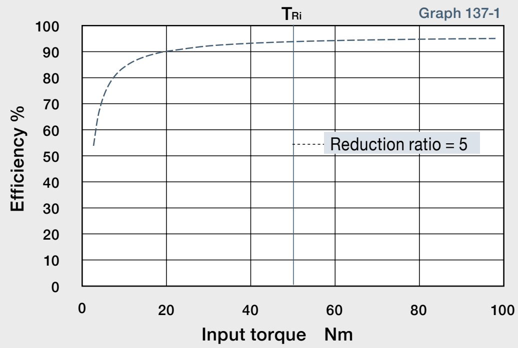


--- Gearhead (standard item) ...... Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

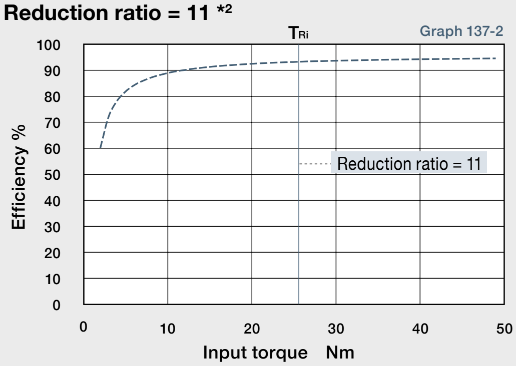
*1 Only one line is shown because the difference between the gearhead and a bearing assembled on the input side is small.

Size 50 : Gearhead HPGP

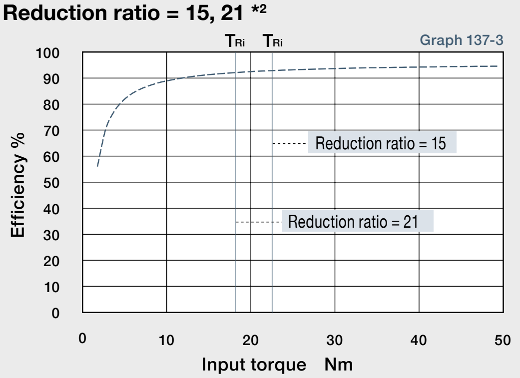
Reduction ratio = 5 *2



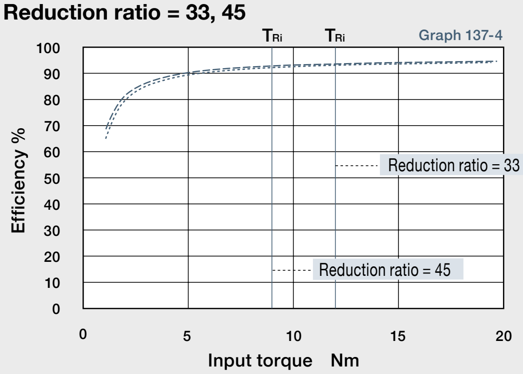
Reduction ratio = 11 *2



Reduction ratio = 15, 21 *2



Reduction ratio = 33, 45

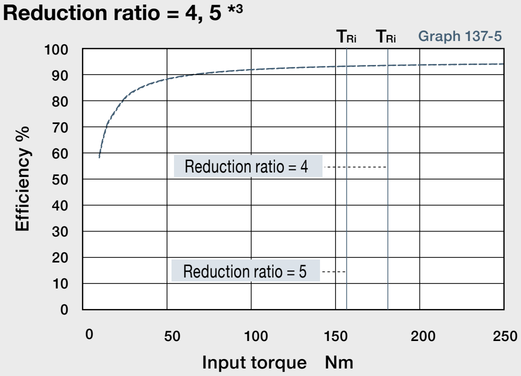


--- Gearhead (standard item) ······ Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

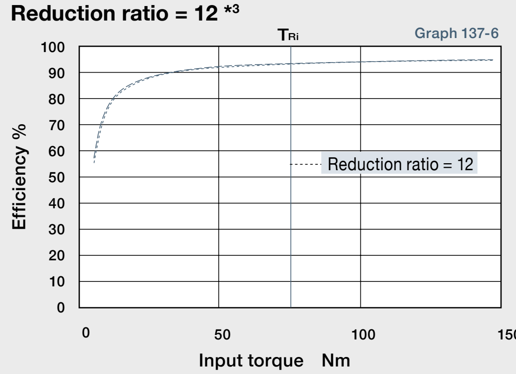
*2 Only one line is shown because the difference between the gearhead and a bearing assembled on the input side is small.

Size 65 : Gearhead HPGP

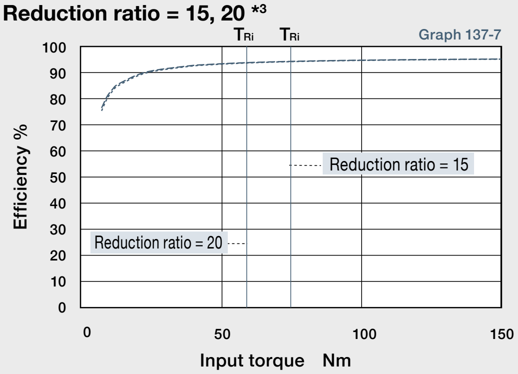
Reduction ratio = 4, 5 *3



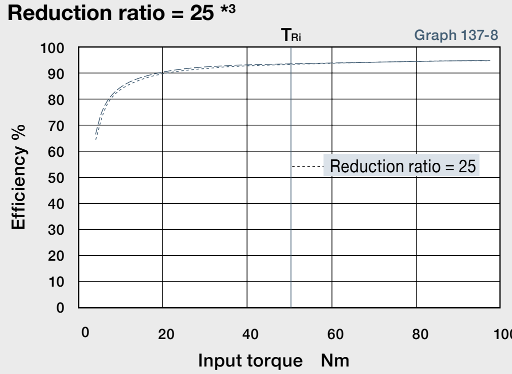
Reduction ratio = 12 *3



Reduction ratio = 15, 20 *3



Reduction ratio = 25 *3



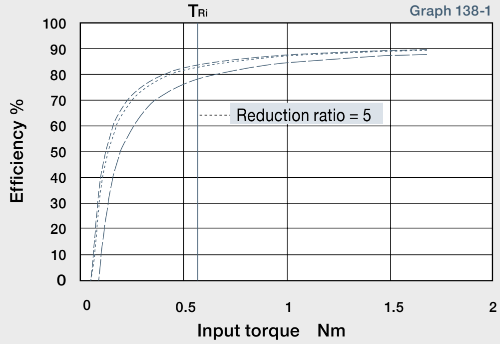
--- Gearhead (standard item) ······ Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

*3 Only one line is shown because the difference between the gearhead and a bearing assembled on the input side is small.

Size 11 : Gearhead & Input Shaft Unit HPG

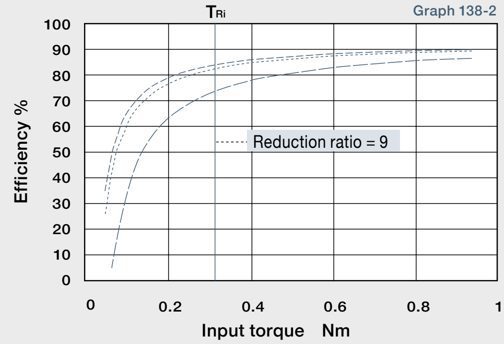
Reduction ratio = 5

Graph 138-1



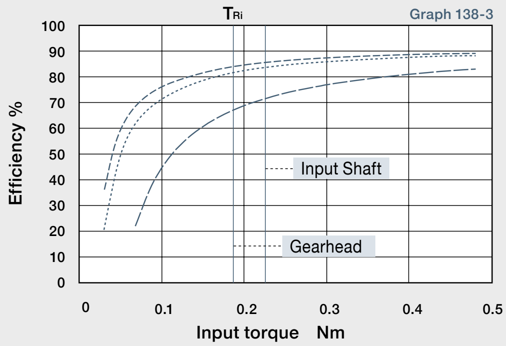
Reduction ratio = 9

Graph 138-2



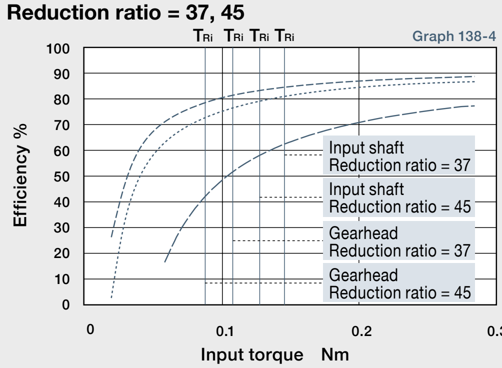
Reduction ratio = 21

Graph 138-3



Reduction ratio = 37, 45

Graph 138-4

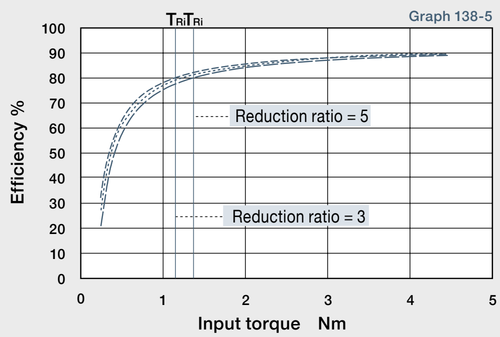


--- Gearhead (standard item) ······ Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

Size 14 : Gearhead & Input Shaft Unit HPG

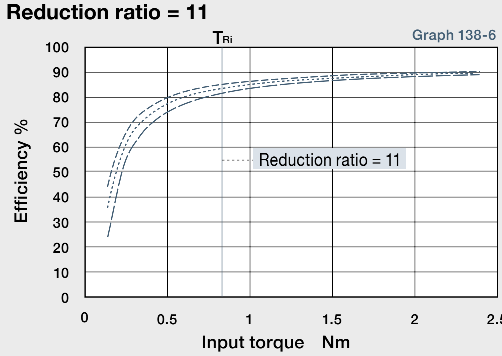
Reduction ratio = 3, 5

Graph 138-5



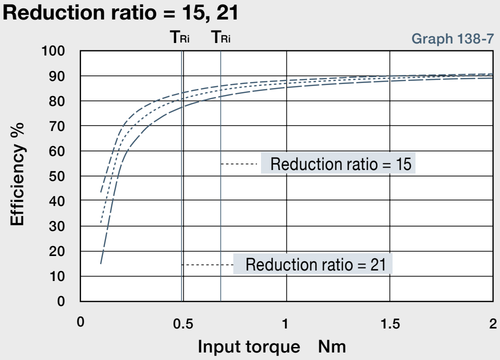
Reduction ratio = 11

Graph 138-6



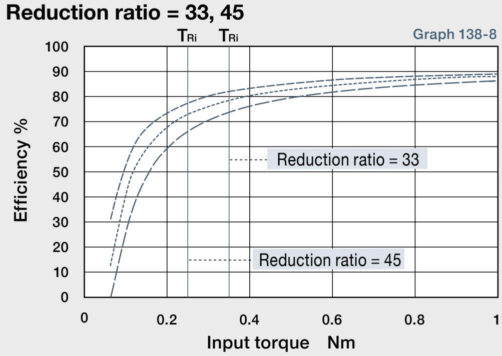
Reduction ratio = 15, 21

Graph 138-7



Reduction ratio = 33, 45

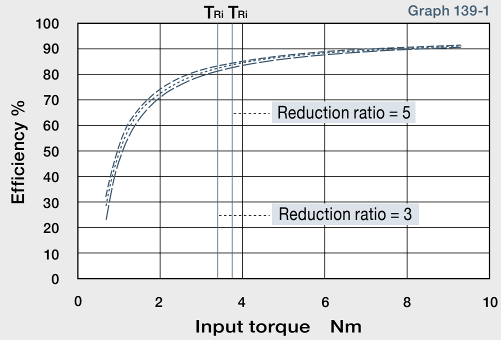
Graph 138-8



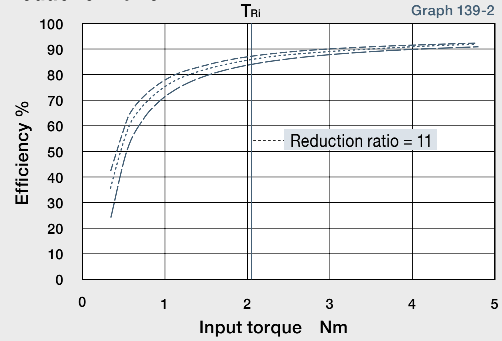
--- Gearhead (standard item) ······ Gearhead with D bearing (double sealed) TRi Input torque corresponding to output torque

Size 20 : Gearhead & Input Shaft Unit HPG

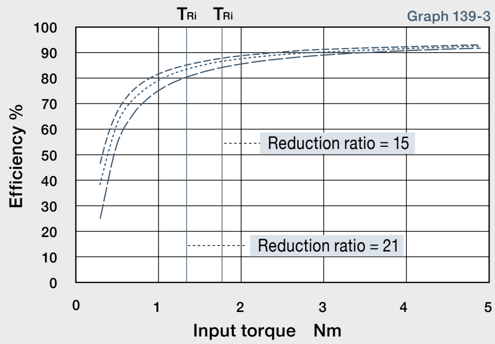
Reduction ratio = 3, 5



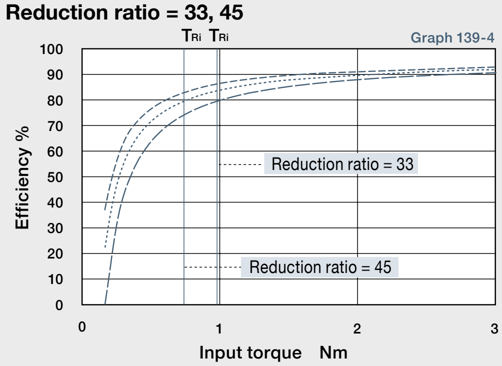
Reduction ratio = 11



Reduction ratio = 15, 21



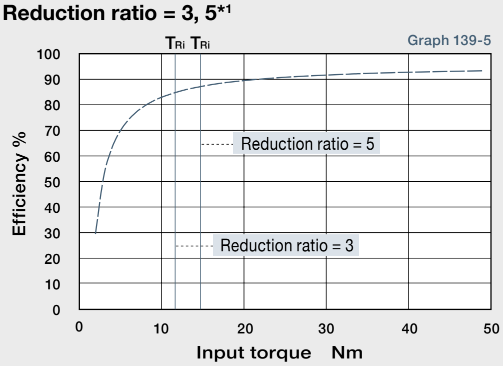
Reduction ratio = 33, 45



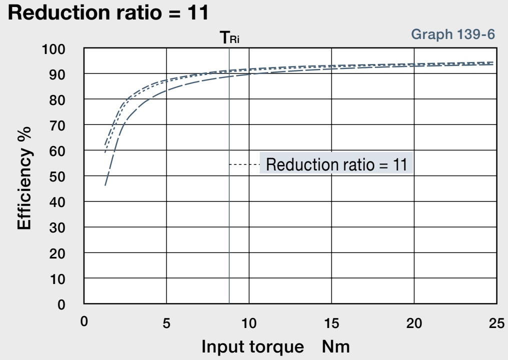
--- Gearhead (standard item) ······ Gearhead with D bearing (double sealed) —— Input Shaft TRi Input torque corresponding to output torque

Size 32 : Gearhead & Input Shaft Unit HPG

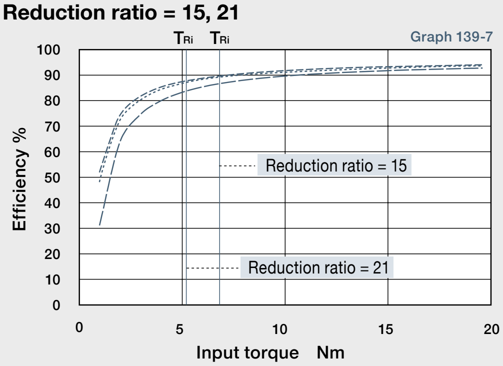
Reduction ratio = 3, 5*1



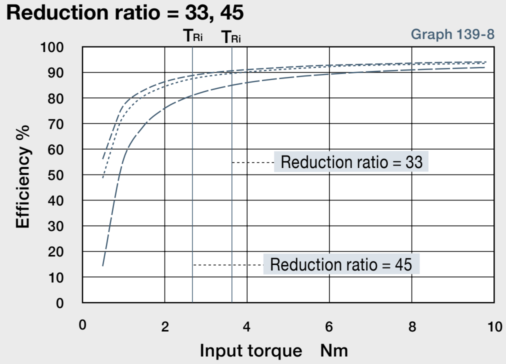
Reduction ratio = 11



Reduction ratio = 15, 21



Reduction ratio = 33, 45

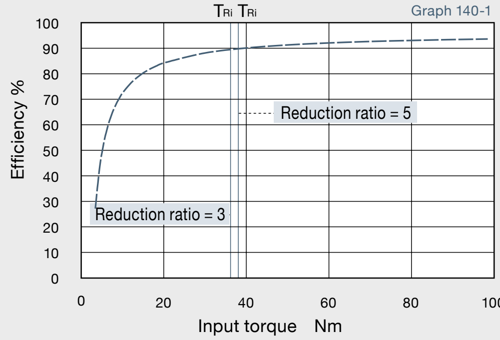


--- Gearhead (standard item) ······ Gearhead with D bearing (double sealed) —— Input Shaft TRi Input torque corresponding to output torque

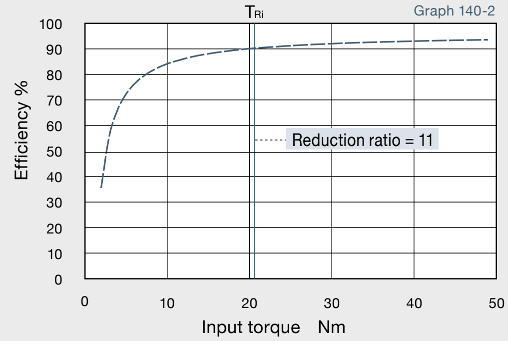
*1 Only one line is shown because the difference between the gearhead and a bearing assembled on the input side is small.

Size 50 : Gearhead & Input Shaft Unit HPG

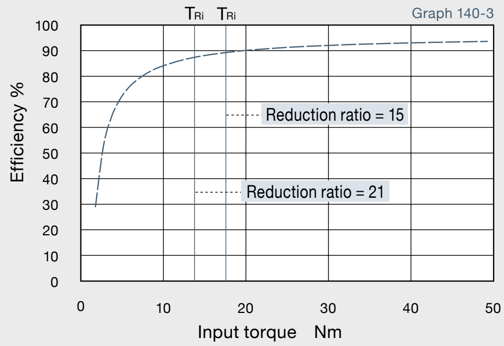
Reduction ratio = 3, 5*2



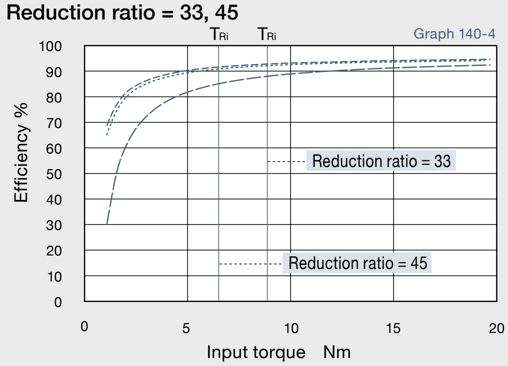
Reduction ratio = 11*2



Reduction ratio = 15, 21*2



Reduction ratio = 33, 45

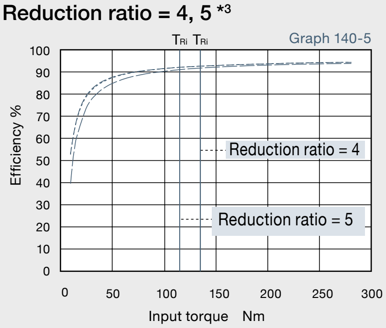


Gearhead (standard item) Gearhead with D bearing (double sealed) Input Shaft TRi Input torque corresponding to output torque

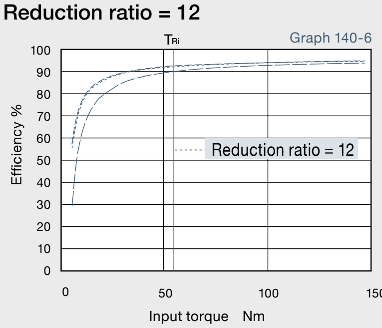
*2 Only one line is shown because the difference between the gearhead and a bearing assembled on the input side is small.

Size 65 : Gearhead & Input Shaft Unit HPG

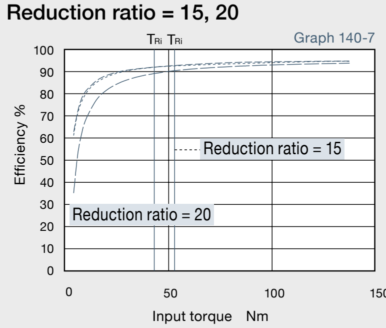
Reduction ratio = 4, 5 *3



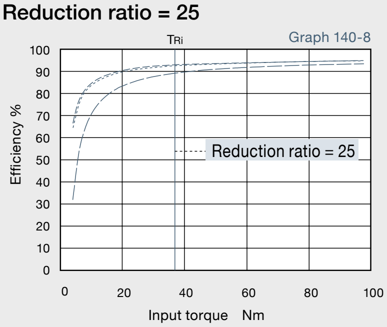
Reduction ratio = 12



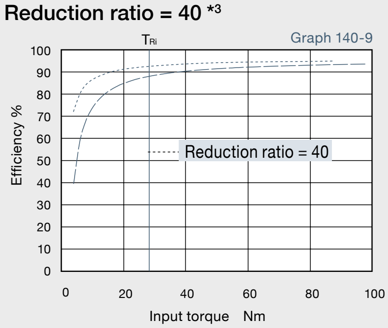
Reduction ratio = 15, 20



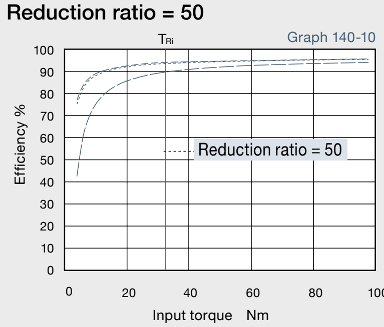
Reduction ratio = 25



Reduction ratio = 40 *3



Reduction ratio = 50

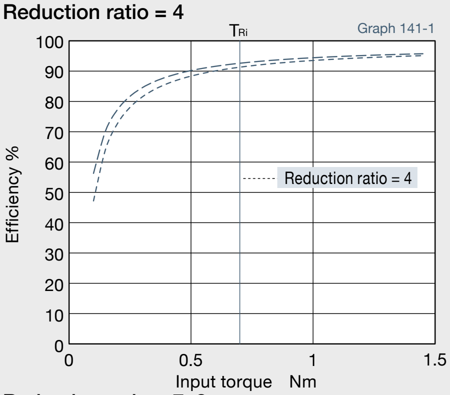


Gearhead (standard item) Gearhead with D bearing (double sealed) Input Shaft TRi Input torque corresponding to output torque

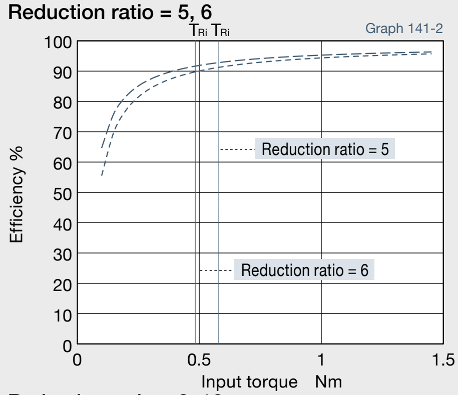
*3 Only one line is shown because the difference between the gearhead and a bearing assembled on the input side is small.

Size 11 : Gearhead HPG-Helical

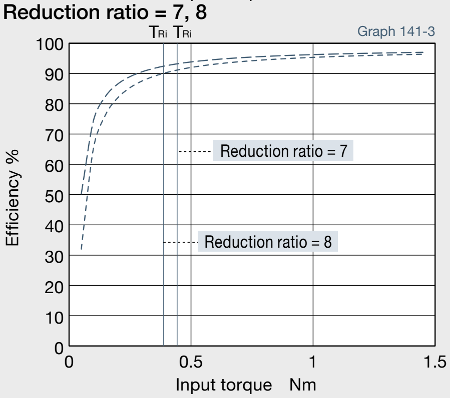
Reduction ratio = 4



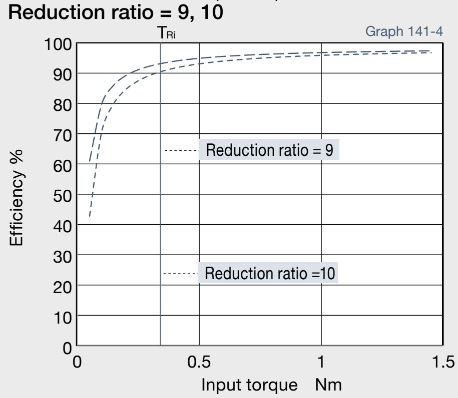
Reduction ratio = 5, 6



Reduction ratio = 7, 8



Reduction ratio = 9, 10



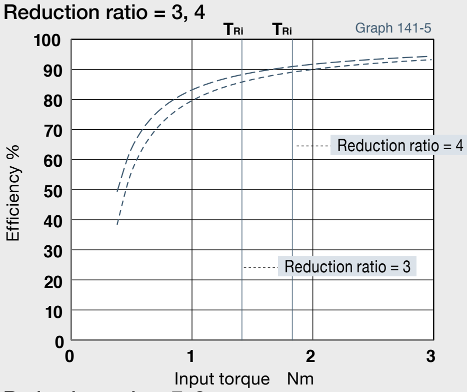
Gearhead with Z bearing (Double sheilded)

Gearhead with D bearing (double sealed)

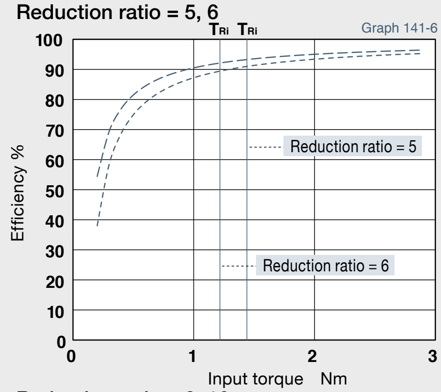
T_{Ri} Input torque corresponding to output torque

Size 14 : Gearhead HPG-Helical

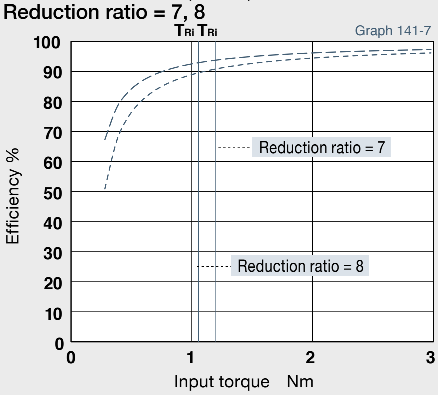
Reduction ratio = 3, 4



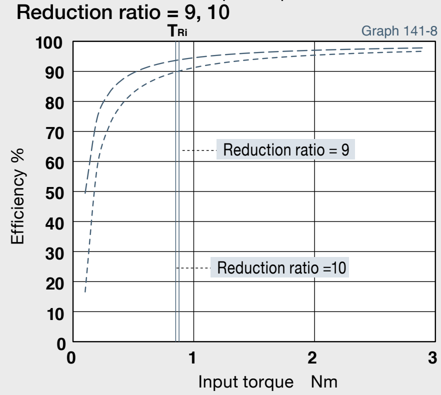
Reduction ratio = 5, 6



Reduction ratio = 7, 8



Reduction ratio = 9, 10



Gearhead with Z bearing (Double sheilded)

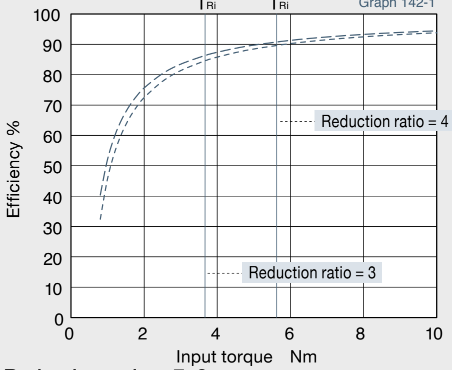
Gearhead with D bearing (double sealed)

T_{Ri} Input torque corresponding to output torque

Size 20 : Gearhead **HPG-Helical**

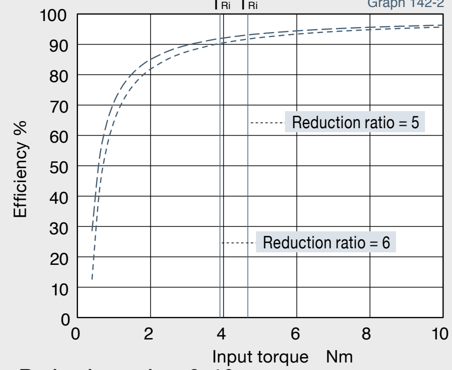
Reduction ratio = 3, 4

Graph 142-1



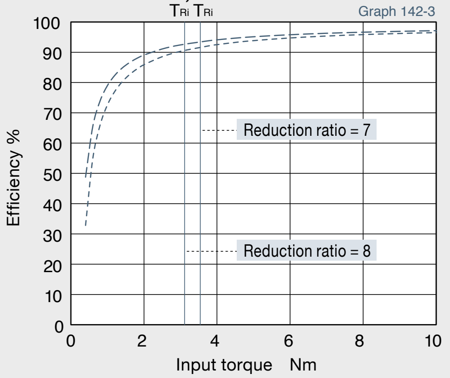
Reduction ratio = 5, 6

Graph 142-2



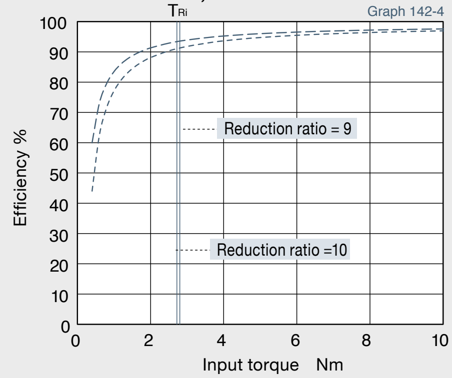
Reduction ratio = 7, 8

Graph 142-3



Reduction ratio = 9, 10

Graph 142-4



--- Gearhead with Z bearing (Double sheilded)

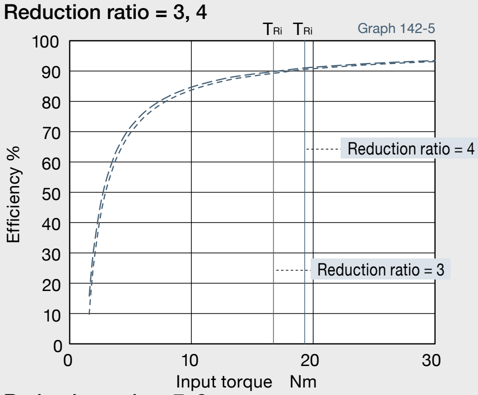
······ Gearhead with D bearing (double sealed)

T_{Ri} Input torque corresponding to output torque

Size 32 : Gearhead **HPG-Helical**

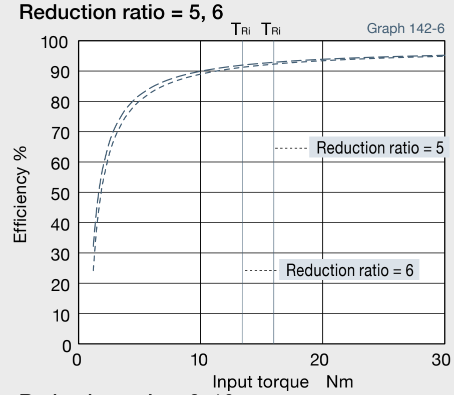
Reduction ratio = 3, 4

Graph 142-5



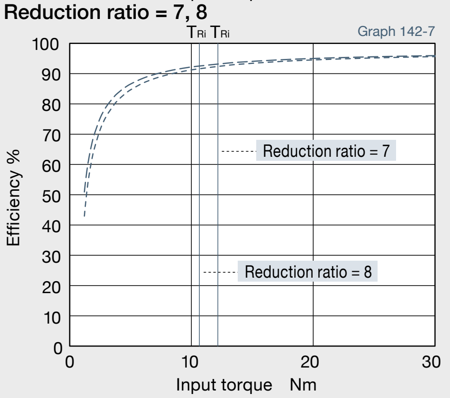
Reduction ratio = 5, 6

Graph 142-6



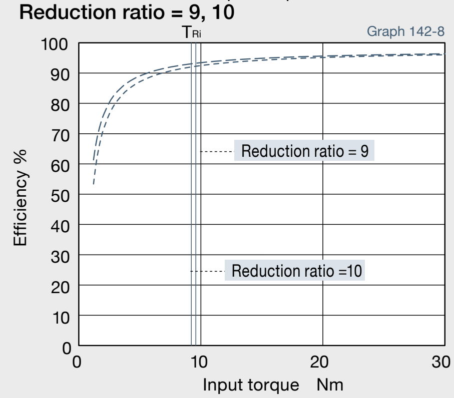
Reduction ratio = 7, 8

Graph 142-7



Reduction ratio = 9, 10

Graph 142-8



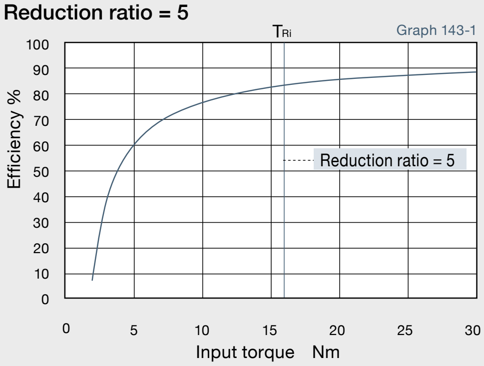
--- Gearhead with Z bearing (Double sheilded)

······ Gearhead with D bearing (double sealed)

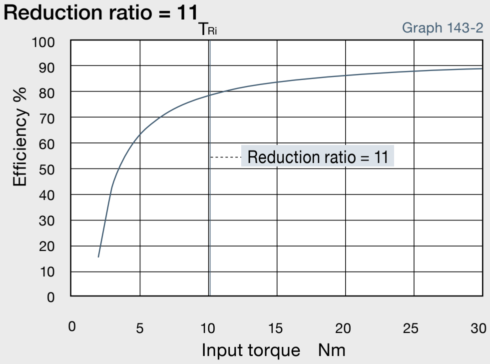
T_{Ri} Input torque corresponding to output torque

Size 32 RA3 : Right Angle Gearhead HPG

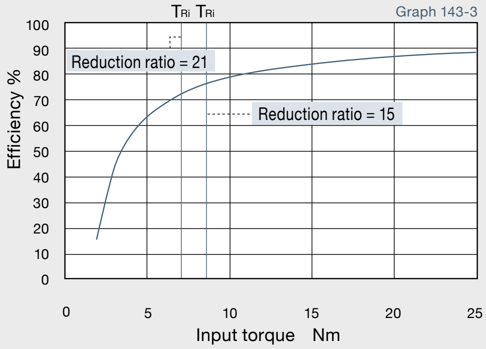
Reduction ratio = 5



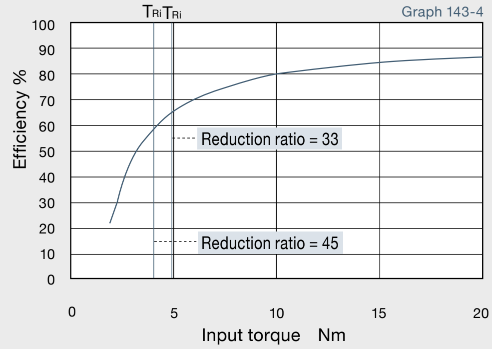
Reduction ratio = 11



Reduction ratio = 15, 21



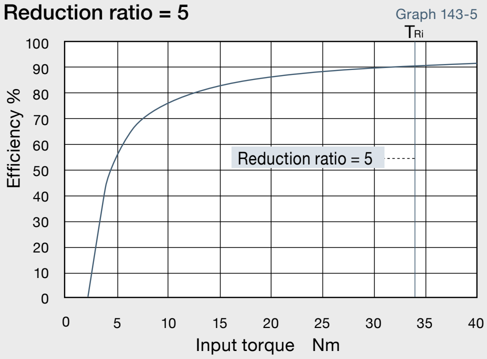
Reduction ratio = 33, 45



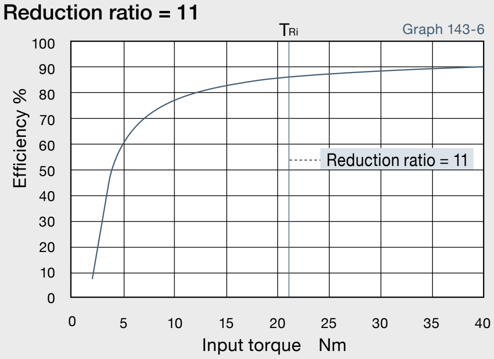
T_{Ri} Input torque corresponding to output torque

Size 50 RA3 : Right Angle Gearhead HPG

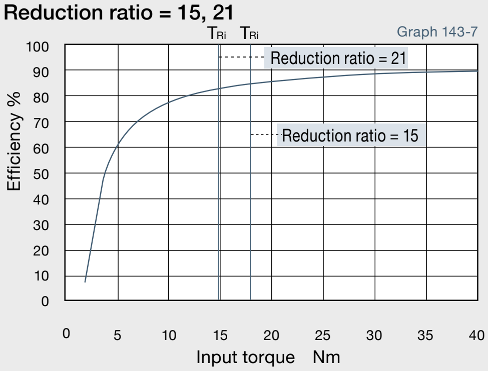
Reduction ratio = 5



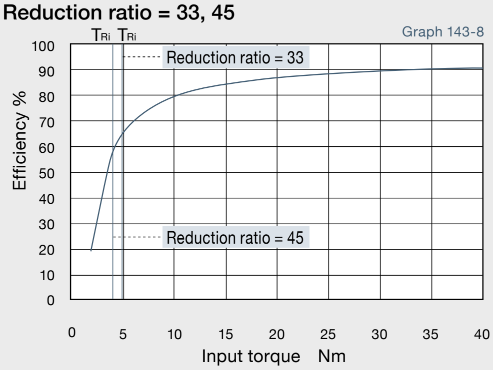
Reduction ratio = 11



Reduction ratio = 15, 21



Reduction ratio = 33, 45

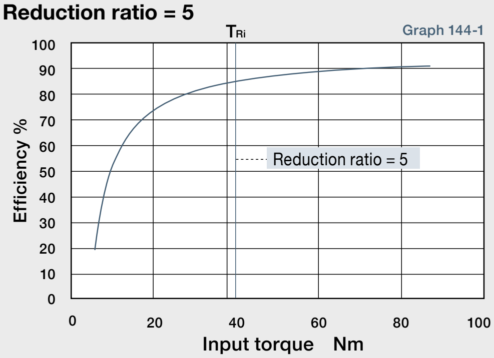


T_{Ri} Input torque corresponding to output torque

Size 50 RA5 : Right Angle Gearhead HPG

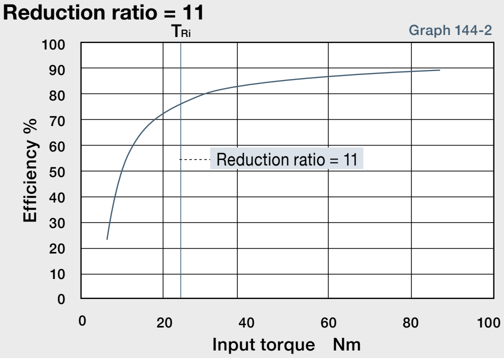
Reduction ratio = 5

Graph 144-1



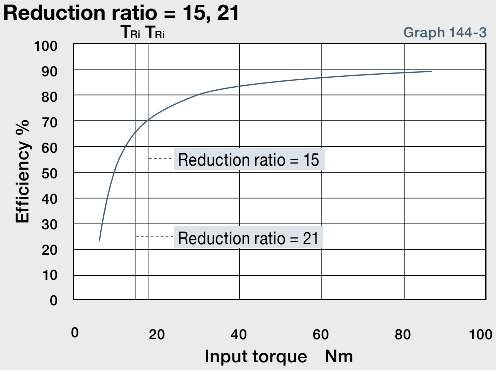
Reduction ratio = 11

Graph 144-2



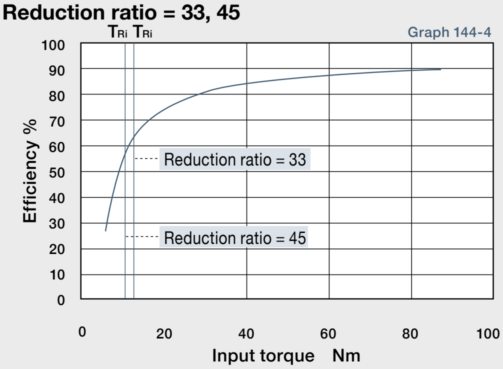
Reduction ratio = 15, 21

Graph 144-3



Reduction ratio = 33, 45

Graph 144-4

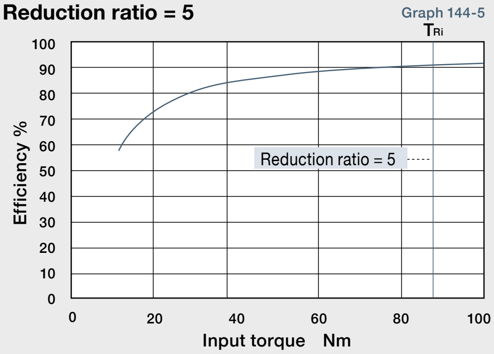


TRi Input torque corresponding to output torque

Size 65 RA5 : Right Angle Gearhead HPG

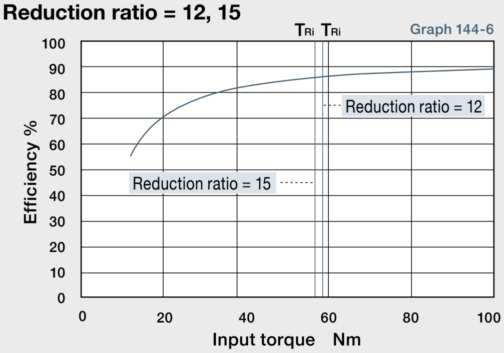
Reduction ratio = 5

Graph 144-5



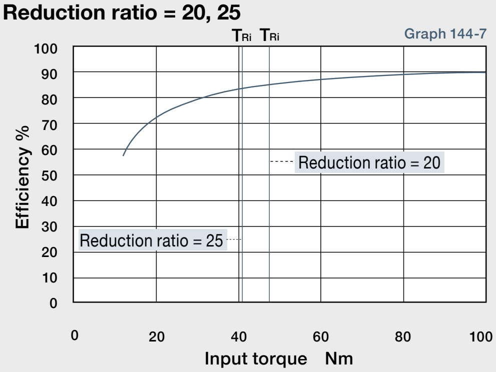
Reduction ratio = 12, 15

Graph 144-6



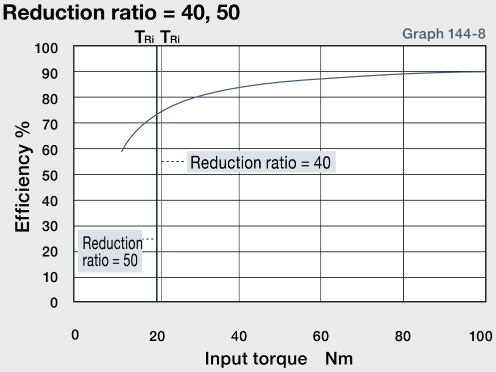
Reduction ratio = 20, 25

Graph 144-7



Reduction ratio = 40, 50

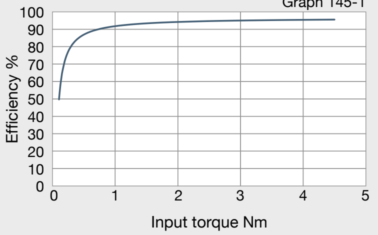
Graph 144-8



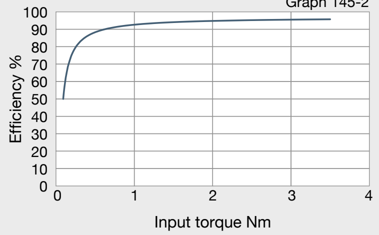
TRi Input torque corresponding to output torque

■ Size 11 HPN

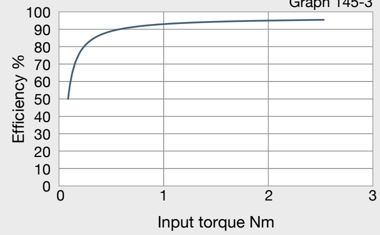
[Reduction ratio = 4]
Graph 145-1



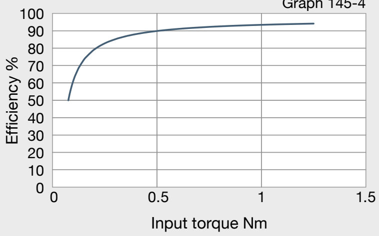
[Reduction ratio = 5]
Graph 145-2



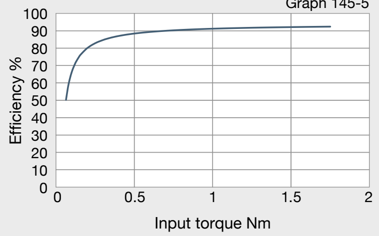
[Reduction ratio = 7]
Graph 145-3



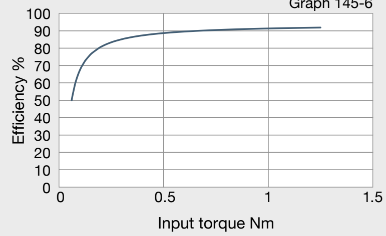
[Reduction ratio = 10]
Graph 145-4



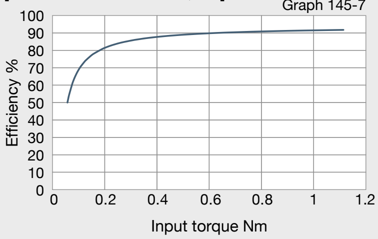
[Reduction ratio = 15]
Graph 145-5



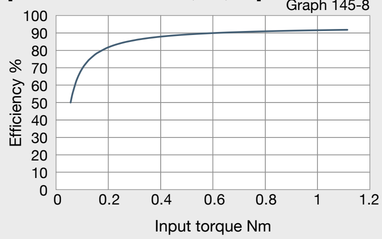
[Reduction ratio = 20, 25]
Graph 145-6



[Reduction ratio = 30, 35]
Graph 145-7

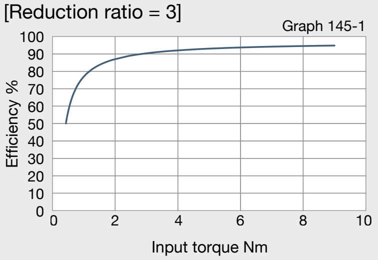


[Reduction ratio = 40, 45, 50]
Graph 145-8

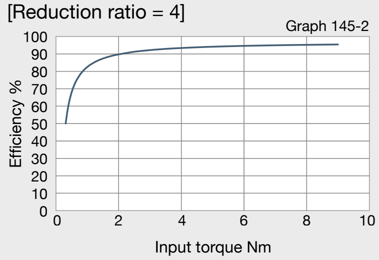


■ Size 14 HPN

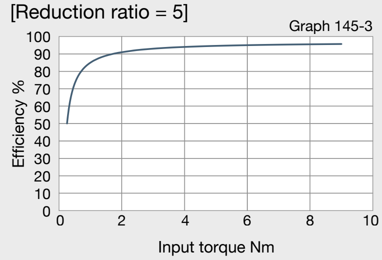
[Reduction ratio = 3]
Graph 145-1



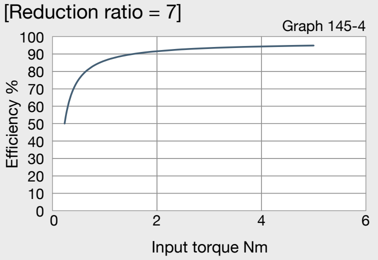
[Reduction ratio = 4]
Graph 145-2



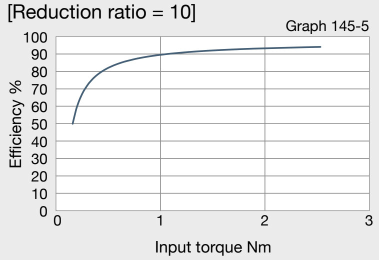
[Reduction ratio = 5]
Graph 145-3



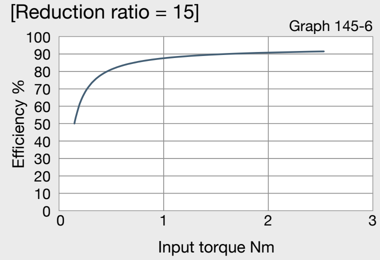
[Reduction ratio = 7]
Graph 145-4



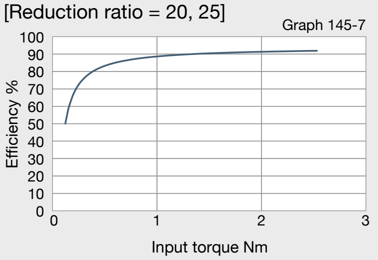
[Reduction ratio = 10]
Graph 145-5



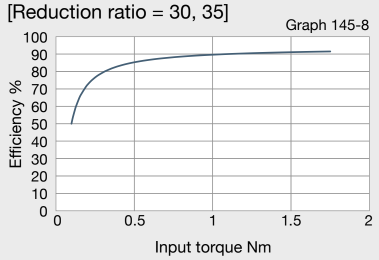
[Reduction ratio = 15]
Graph 145-6



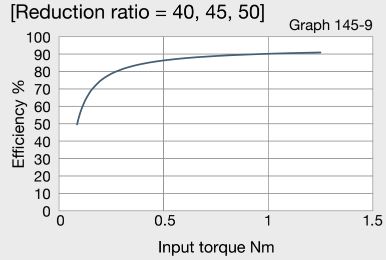
[Reduction ratio = 20, 25]
Graph 145-7



[Reduction ratio = 30, 35]
Graph 145-8

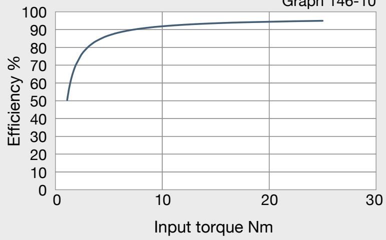


[Reduction ratio = 40, 45, 50]
Graph 145-9

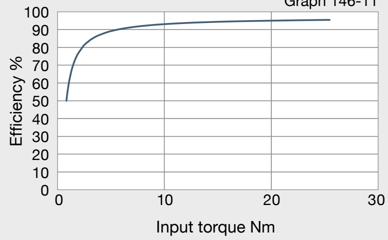


■ Size 20 HPN

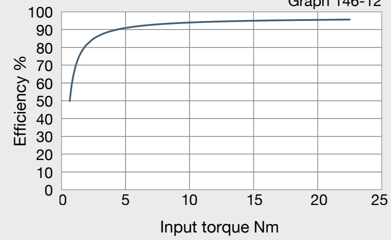
[Reduction ratio = 3]
Graph 146-10



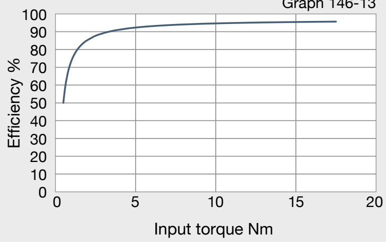
[Reduction ratio = 4]
Graph 146-11



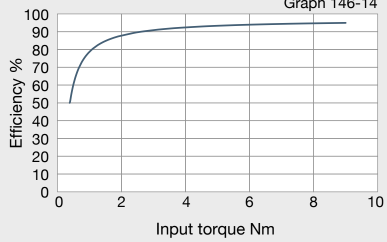
[Reduction ratio = 5]
Graph 146-12



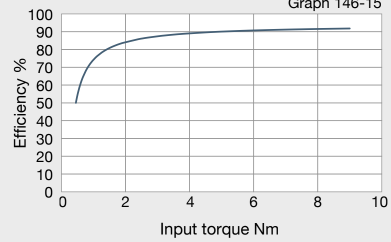
[Reduction ratio = 7]
Graph 146-13



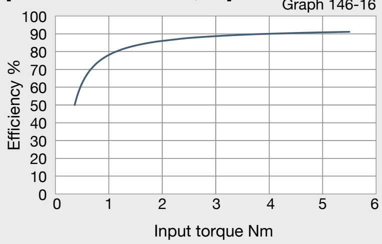
[Reduction ratio = 10]
Graph 146-14



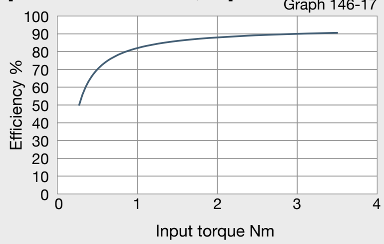
[Reduction ratio = 15]
Graph 146-15



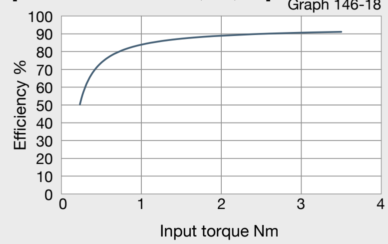
[Reduction ratio = 20, 25]
Graph 146-16



[Reduction ratio = 30, 35]
Graph 146-17

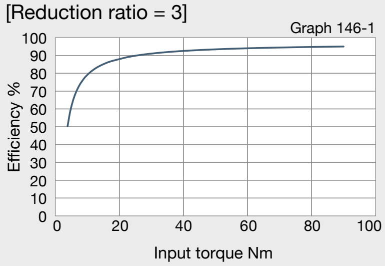


[Reduction ratio = 40, 45, 50]
Graph 146-18

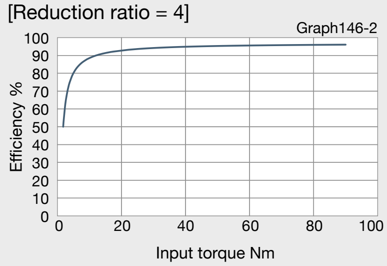


■ Size 32 HPN

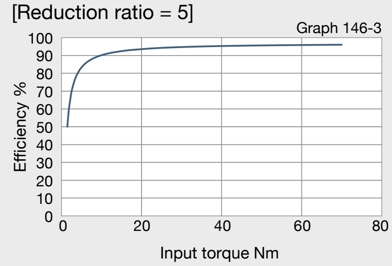
[Reduction ratio = 3]
Graph 146-1



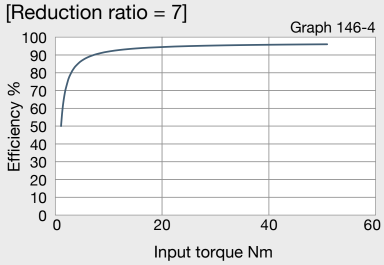
[Reduction ratio = 4]
Graph146-2



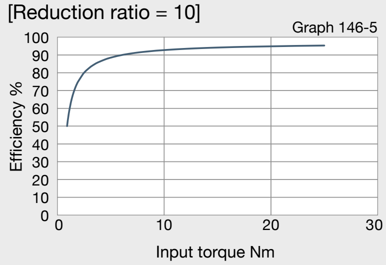
[Reduction ratio = 5]
Graph 146-3



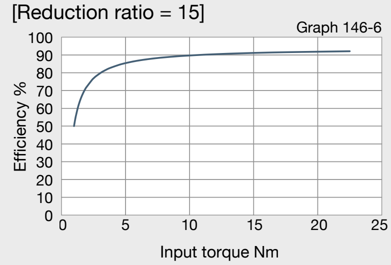
[Reduction ratio = 7]
Graph 146-4



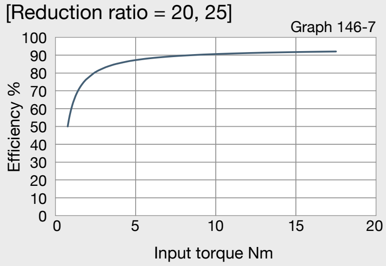
[Reduction ratio = 10]
Graph 146-5



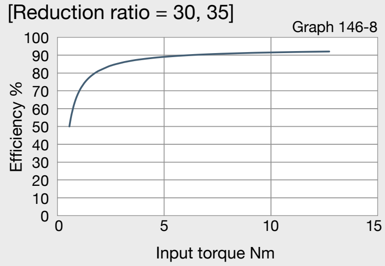
[Reduction ratio = 15]
Graph 146-6



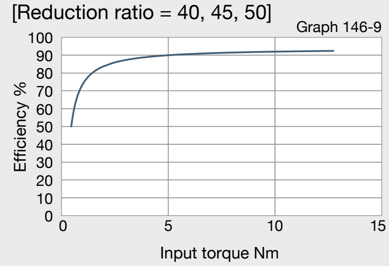
[Reduction ratio = 20, 25]
Graph 146-7



[Reduction ratio = 30, 35]
Graph 146-8

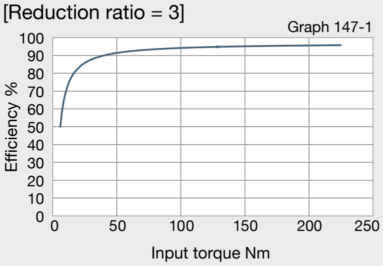


[Reduction ratio = 40, 45, 50]
Graph 146-9

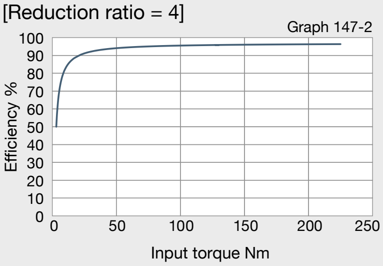


■ Size 40 **HPN**

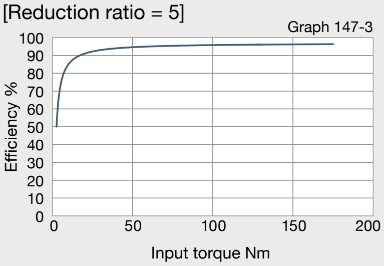
[Reduction ratio = 3]



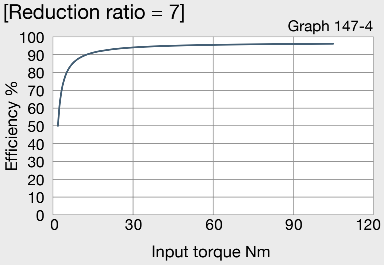
[Reduction ratio = 4]



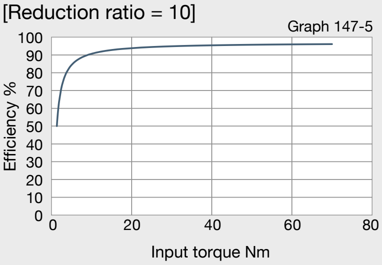
[Reduction ratio = 5]



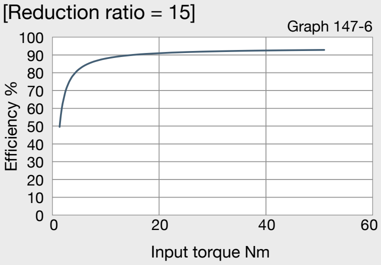
[Reduction ratio = 7]



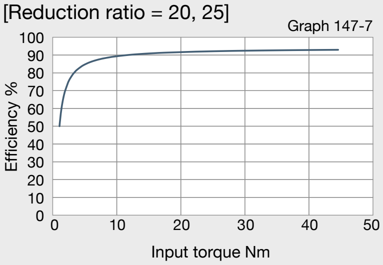
[Reduction ratio = 10]



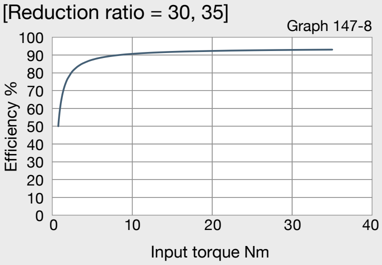
[Reduction ratio = 15]



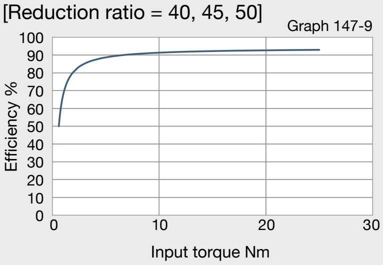
[Reduction ratio = 20, 25]



[Reduction ratio = 30, 35]



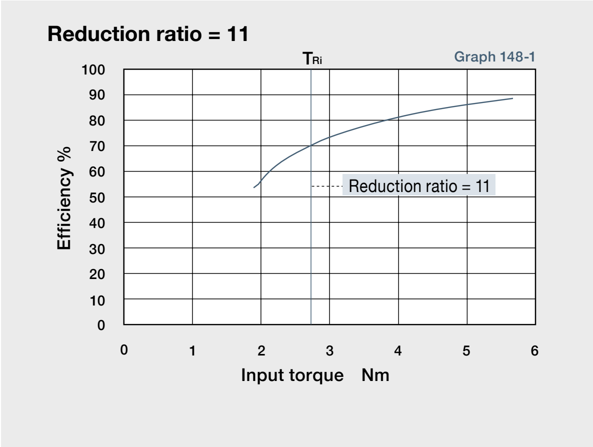
[Reduction ratio = 40, 45, 50]



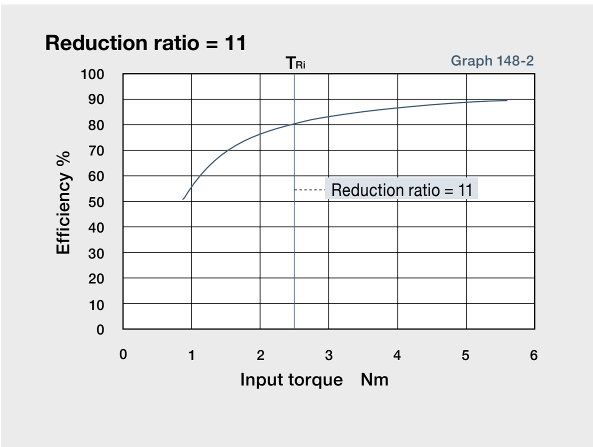
Technical Information / Handling Explanation

HarmonicPlanetary® & HarmonicDrive®

Size 25 : Hollow Shaft Unit HPF



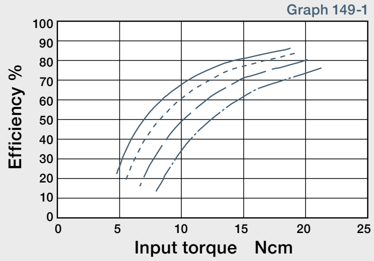
Size 32 : Hollow Shaft Unit HPF



Size 14 : Gearhead CSG-GH CSF-GH

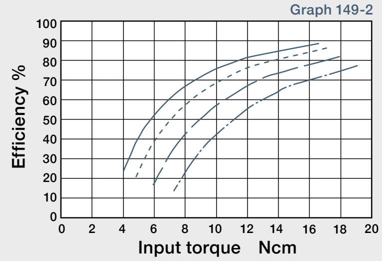
Reduction ratio = 50

Graph 149-1



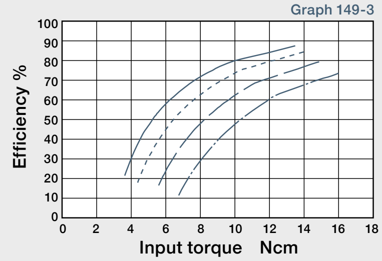
Reduction ratio = 80

Graph 149-2



Reduction ratio = 100

Graph 149-3

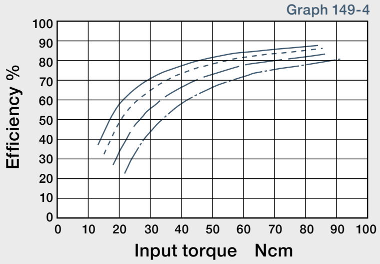


Input rotational speed ——— 500 rpm - - - - - - 1000 rpm —— —— 2000 rpm —·— 3500 rpm

Size 20 : Gearhead CSG-GH CSF-GH

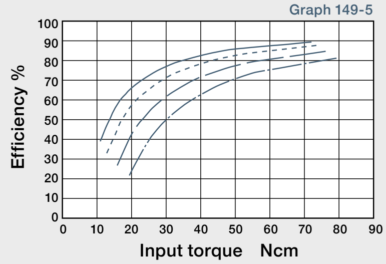
Reduction ratio = 50

Graph 149-4



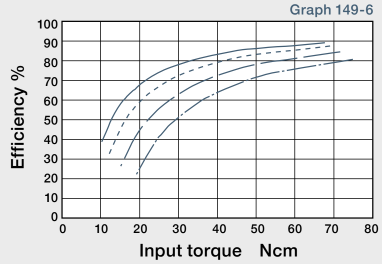
Reduction ratio = 80

Graph 149-5



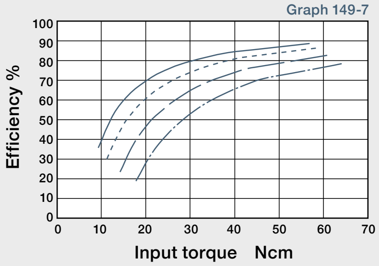
Reduction ratio = 100

Graph 149-6



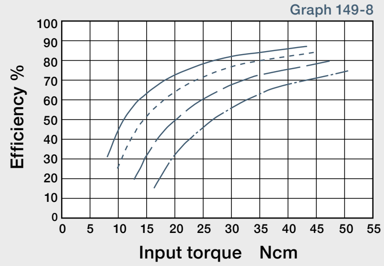
Reduction ratio = 120

Graph 149-7



Reduction ratio = 160

Graph 149-8



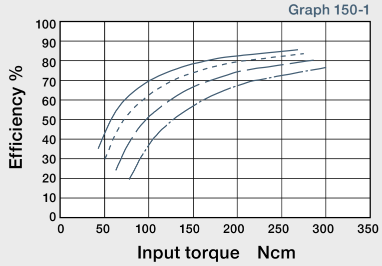
Input rotational speed ——— 500 rpm - - - - - - 1000 rpm —— —— 2000 rpm —·— 3500 rpm

Size 32 : Gearhead

CSG-GH CSF-GH

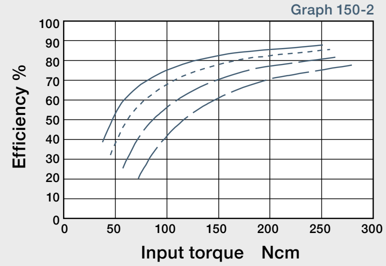
Reduction ratio = 50

Graph 150-1



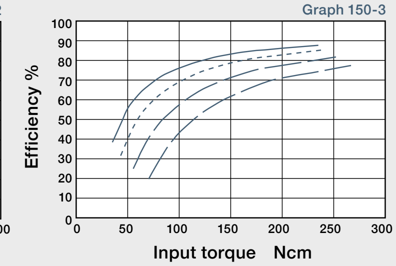
Reduction ratio = 80

Graph 150-2



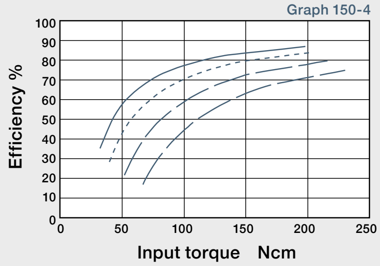
Reduction ratio = 100

Graph 150-3



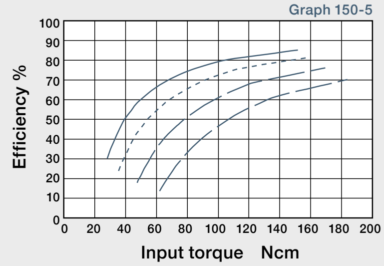
Reduction ratio = 120

Graph 150-4



Reduction ratio = 160

Graph 150-5



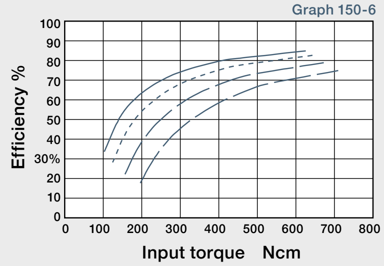
Input rotational speed ——— 500 rpm - - - - - - 1000 rpm —— —— 2000 rpm ——·—— 3500 rpm

Size 45 : Gearhead

CSG-GH CSF-GH

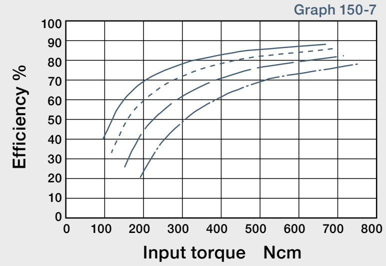
Reduction ratio = 50

Graph 150-6



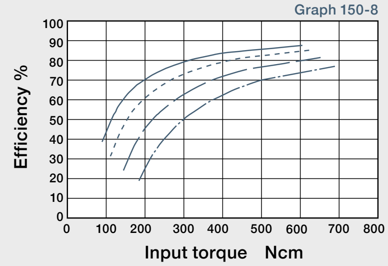
Reduction ratio = 80

Graph 150-7



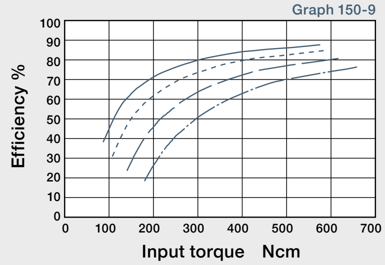
Reduction ratio = 100

Graph 150-8



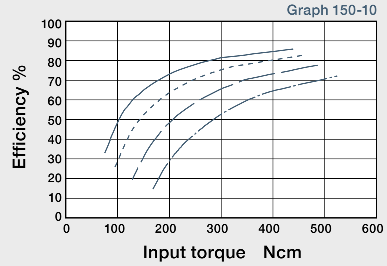
Reduction ratio = 120

Graph 150-9



Reduction ratio = 160

Graph 150-10

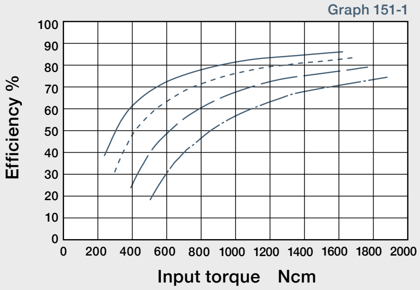


Input rotational speed ——— 500 rpm - - - - - - 1000 rpm —— —— 2000 rpm ——·—— 3500 rpm

Size 65 : Gearhead **CSG-GH** **CSF-GH**

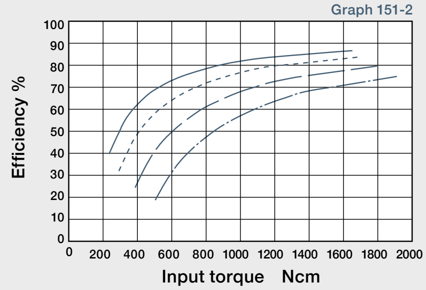
Reduction ratio = 80

Graph 151-1



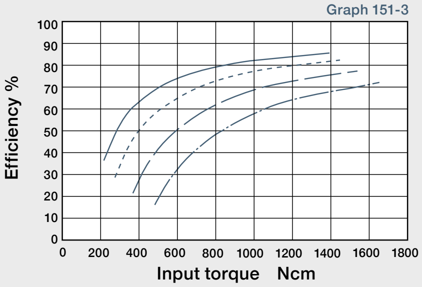
Reduction ratio = 100

Graph 151-2



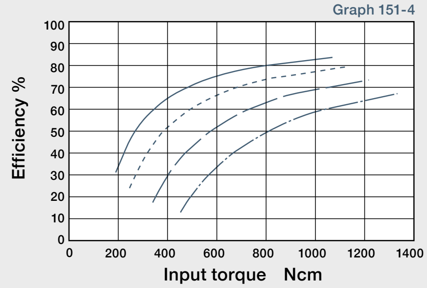
Reduction ratio = 120

Graph 151-3



Reduction ratio = 160

Graph 151-4



Input rotational speed ——— 500 rpm - - - - - - 1000 rpm —— —— 2000 rpm —·— 3500 rpm

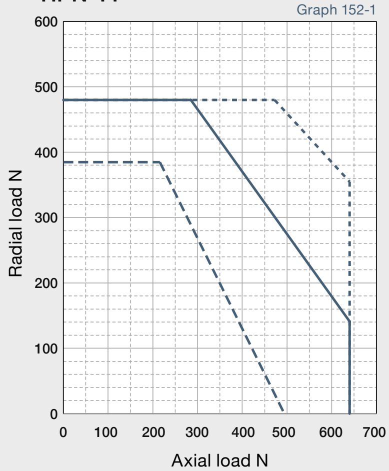
Output Shaft Bearing Load Limits

HPN Series Output Shaft Load Limits are plotted below.

HPN uses deep groove ball bearings to support the output shaft. Please use the curve on the graph for the appropriate load coefficient (fw) that represents the expected operating condition.

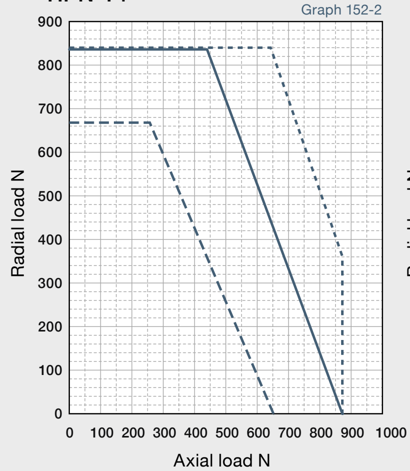
HPN-11

Graph 152-1



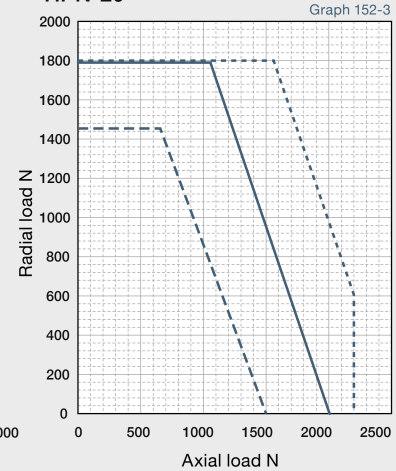
HPN-14

Graph 152-2



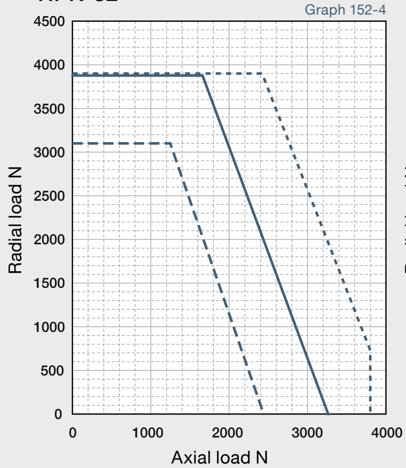
HPN-20

Graph 152-3



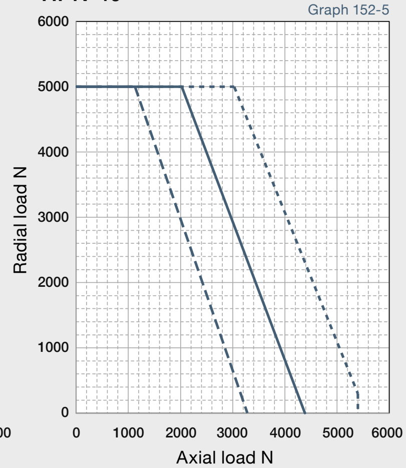
HPN-32

Graph 152-4



HPN-40

Graph 152-5



- - - fw=1
—— fw=1.2
– – fw=1.5

Load coefficient
fw=1~1.2 Smooth operation without impact
fw=1.2~1.5 Standard operation

Output shaft speed - 100 rpm, bearing life is based on 20,000 hours. The load-point is based on shaft center of radial load and axial load.

Output Bearing Specifications and Checking Procedure

HPGP, HPG, HPG Helical, CSF-GH, CSG-GH, HPF, and HPG-U1 are equipped with cross roller bearings. A precision cross roller bearing supports the external load (output flange).
Check the maximum load, moment load, life of the bearing and static safety coefficient to maximize performance.

Checking procedure

(1) Checking the maximum moment load (M *max*)

Calculate the maximum moment load (M_{max}). ▶ Maximum moment load (M_{max}) ≦ Permissible moment (Mc)

(2) Checking the life

Calculate the average radial load (Fr_{av}) and the average axial load (Fa_{av}). ▶ Calculate the radial load coefficient (X) and the axial load coefficient (Y). ▶ Calculate the life and check it.

(3) Checking the static safety coefficient

Calculate the static equivalent radial load coefficient (Po). ▶ Check the static safety coefficient. (fs)

Specification of output bearing

HPGP/HPG Series Tables 153-1, -2 and -3 indicate the cross roller bearing specifications for in-line, right angle and input shaft gears.

Table 153-1

Size	Pitch circle	Offset amount	Basic rated load				Allowable moment load Mc*3		Moment stiffness Km*4	
	dp	R	Basic dynamic load rating C*1		Basic static load rating Co*2		Nm	Kgfm	×10⁴ Nm/rad	Kgfm/arc min
	m	m	N	kgf	N	kgf				
11	0.0275	0.006	3116	318	4087	417	9.50	0.97	0.88	0.26
14	0.0405	0.011	5110	521	7060	720	32.3	3.30	3.0	0.90
20	0.064	0.0115	10600	1082	17300	1765	183	18.7	16.8	5.0
32	0.085	0.014	20500	2092	32800	3347	452	46.1	42.1	12.5
50	0.123	0.019	41600	4245	76000	7755	1076	110	100	29.7
65	0.170	0.023	90600	9245	148000	15102	3900	398	364	108

Table 153-2

Size	Reduction ratio	Allowable radial load*5	Allowable axial load*5
		N	N
11	5	280	430
	(9)	340	510
	21	440	660
	37	520	780
	45	550	830
14	(3)	400	600
	5	470	700
	11	600	890
	15	650	980
	21	720	1080
	33	830	1240
	45	910	1360
20	(3)	840	1250
	5	980	1460
	11	1240	1850
	15	1360	2030
	21	1510	2250
	33	1729	2580
	45	1890	2830

* The ratio specified in parentheses is for the HPG Series.

Table 153-3

Size	Reduction ratio	Allowable radial load*5	Allowable axial load*5
		N	N
32	(3)	1630	2430
	5	1900	2830
	11	2410	3590
	15	2640	3940
	21	2920	4360
	33	3340	4990
	45	3670	5480
50	(3)	3700	5570
	5	4350	6490
	11	5500	8220
	15	6050	9030
	21	6690	9980
	33	7660	11400
	45	8400	12500
65	4	8860	13200
	5	9470	14100
	12	12300	18300
	15	13100	19600
	20	14300	21400
	25	15300	22900
	(40)	17600	26300
	(50)	18900	28200

* The ratio specified in parentheses is for the HPG Series.

〔Note：Table 153-1, -2 and -3 Table 154-1 and -2〕

*1 The basic dynamic load rating means a certain static radial load so that the basic dynamic rated life of the roller bearing is a million rotations.
*2 The basic static load rating means a static load that gives a certain level of contact stress (4kN/mm²) in the center of the contact area between rolling element receiving the maximum load and orbit.
*3 The allowable moment load is a maximum moment load applied to the bearing. Within the allowable range, basic performance is maintained and the bearing is operable. Check the bearing life based on the calculations shown on the next page.
*4 The value of the moment stiffness is the average value.
*5 The allowable radial load and allowable axial load are the values that satisfy the life of a speed reducer when a pure radial load or an axial load applies to the main bearing. (Lr + R = 0 mm for radial load and La = 0 mm for axial load) If a compound load applies, refer to the calculations shown on the next page.

CSG-GH/CSF-GH Series Table 154-1 indicates the specifications for cross roller bearing.

Table 154-1

Size	Pitch circle dp	Offset amount R	Basic load rating: Basic dynamic load rating C*1		Basic load rating: Basic static load rating Co*2		Allowable moment load Mc*3		Moment stiffness Km*4		Allowable radial load*5	Allowable axial load*5
	m	m	N	kgf	N	kgf	Nm	kgfm	×10⁴ Nm/rad	kgfm/arc min	N	N
14	0.0405	0.011	5110	521	7060	720	27	2.76	3.0	0.89	732	1093
20	0.064	0.0115	10600	1082	17300	1765	145	14.8	17	5.0	1519	2267
32	0.085	0.014	20500	2092	32800	3347	258	26.3	42	12	2938	4385
45	0.123	0.019	41600	4245	76000	7755	797	81.3	100	30	5962	8899
65	0.170	0.0225	81600	8327	149000	15204	2156	220	323	96	11693	17454

HPF Series Table 154-2 indicates the specifications for cross roller bearing.

Table 154-2

Size	Pitch circle dp	Offset amount R	Basic load rating: Basic dynamic load rating C*1		Basic load rating: Basic static load rating Co*2		Allowable moment load Mc*3		Moment stiffness Km*4		Allowable radial load*5	Allowable axial load*5
	m	m	N	kgf	N	kgf	Nm	kgfm	×10⁴ Nm/rad	kgfm/arc min	N	N
25	0.085	0.0153	11400	1163	20300	2071	410	41.8	37.9	11.3	1330	1990
32	0.1115	0.015	22500	2296	39900	4071	932	95	86.1	25.7	2640	3940

〔Note：Table 153-1, -2 and -3　Table 154-1 and -2〕

*1 The basic dynamic load rating means a certain static radial load so that the basic dynamic rated life of the roller bearing is a million rotations.
*2 The basic static load rating means a static load that gives a certain level of contact stress (4kN/mm^2) in the center of the contact area between rolling element receiving the maximum load and orbit.
*3 The allowable moment load is a maximum moment load applied to the bearing. Within the allowable range, basic performance is maintained and the bearing is operable. Check the bearing life based on the calculations shown on the next page.
*4 The value of the moment stiffness is the average value.
*5 The allowable radial load and allowable axial load are the values that satisfy the life of a speed reducer when a pure radial load or an axial load applies to the main bearing. (Lr + R = 0 mm for radial load and La = 0 mm for axial load) If a compound load applies, refer to the calculations shown on the next page.

How to calculate the maximum moment load

HPGP | HPG | CSG-GH | CSF-GH | HPF

Maximum moment load (M_{max}) is obtained as follows.
Make sure that $M_{max} \leqq Mc$.

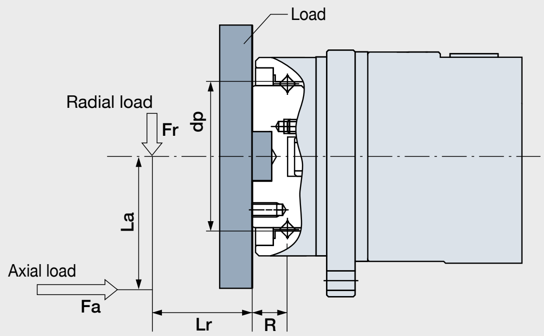
Formula 155-1

$$M_{max} = Fr_{max}(Lr+R) + Fa_{max}\, La$$

Fr_{max}	Max. radial load	N (kgf)	See Fig. 155-1.
Fa_{max}	Max. axial load	N (kgf)	See Fig. 155-1.
Lr, La	—	m	See Fig. 155-1.
R	Offset amount	m	See Fig. 155-1. See "Output Bearing Specifications" of each series, p.153 & 154

Figure 155-1

External load influence diagram



How to calculate the radial and the axial load coefficient

HPGP | HPG | CSG-GH | CSF-GH | HPF

The radial load coefficient (X) and the axial load coefficient (Y)

Formula 155-2

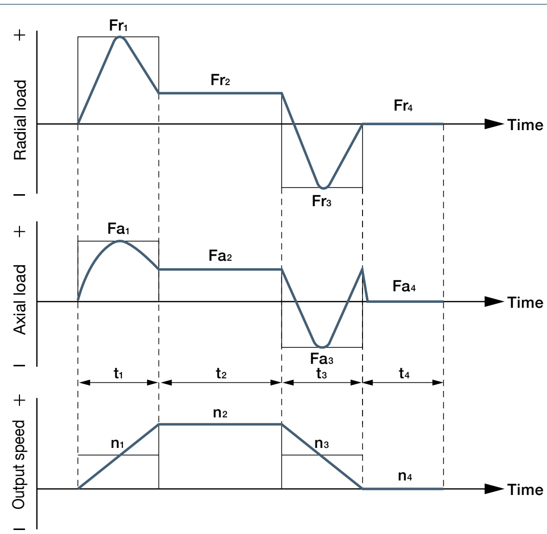
Formula	X	Y
$\dfrac{Fa_{av}}{Fr_{av}+2(Fr_{av}(Lr+R)+Fa_{av}\cdot La)/dp} \leqq 1.5$	1	0.45
$\dfrac{Fa_{av}}{Fr_{av}+2(Fr_{av}(Lr+R)+Fa_{av}\cdot La)/dp} > 1.5$	0.67	0.67

Fr_{av}	Average radial load	N (kgf)	See "How to calculate the average load below."
Fa_{av}	Average axial load	N (kgf)	See "How to calculate the average load below."
Lr, La	—	m	See Fig. 155-1.
R	Offset amount	m	See Fig. 155-1. See "Output Bearing Specifications" of each series, p. 153 & 154.
dp	Circlar pitch of roller	m	See Fig. 155-1. See "Output Bearing Specifications" of each series, p. 153 & 154.

How to calculate the average load (Average radial load, average axial load, average output speed)

HPGP | HPG | CSG-GH | CSF-GH | HPF

If the radial load and the axial load fluctuate, they should be converted into the average load to check the life of the cross roller bearing.



How to obtain the average radial load (Fr_{av})

Formula 155-3

$$Fr_{av} = \sqrt[10/3]{\frac{n_1 t_1 (|Fr_1|)^{10/3} + n_2 t_2 (|Fr_2|)^{10/3} \cdots n_n t_n (|Fr_n|)^{10/3}}{n_1 t_1 + n_2 t_2 + \cdots + n_n t_n}}$$

Note that the maximum radial load within the t_1 section is Fr_1 and the maximum radial load within the t_3 section is Fr_3.

How to obtain the average axial load (Fa_{av})

Formula 155-4

$$Fa_{av} = \sqrt[10/3]{\frac{n_1 t_1 (|Fa_1|)^{10/3} + n_2 t_2 (|Fa_2|)^{10/3} \cdots n_n t_n (|Fa_n|)^{10/3}}{n_1 t_1 + n_2 t_2 + \cdots + n_n t_n}}$$

Note that the maximum axial load within the t_1 section is Fa_1 and the maximum axial load within the t_3 section is Fa_3.

How to obtain the average output speed (N_{av})

Formula 155-5

$$N_{av} = \frac{n_1 t_1 + n_2 t_2 + \cdots + n_n t_n}{t_1 + t_2 + \cdots + t_n}$$

How to calculate the life

HPGP HPG CSG-GH CSF-GH HPF

Calculate the life of the cross roller bearing using Formula 156-1. You can obtain the dynamic equivalent load (Pc) using Formula 156-2.

Formula 156-1

$$L_{10}=\frac{10^6}{60\times N_{av}}\times\left(\frac{C}{fw\cdot Pc}\right)^{10/3}$$

L_{10}	Life	hour	—
N_{av}	Ave. output speed	rpm	See "How to calculate the ave. load."
C	Basic dynamic load rating	N (kgf)	See "Output Bearing Specs."
Pc	Dynamic equivalent load	N (kgf)	See Formula 156-2.
fw	Load coefficient	—	See Table 156-1.

Formula 156-2

$$Pc=X\cdot\left(Fr_{av}+\frac{2(Fr_{av}(Lr+R)+Fa_{av}\cdot La)}{dp}\right)+Y\cdot Fa_{av}$$

Fr_{av}	Average radial load	N (kgf)	See "How to calculate the ave. load."
Fa_{av}	Average axial load	N (kgf)	
dp	Pitch Circle of roller	m	See "Output Bearing Specs."
X	Radial load coefficient	-	See "How to calculate the radial load coefficient and the axial load coefficient."
Y	Axial load coefficient	-	
Lr, La	—	m	See Figure 155-1. See "External load influence diagram."
R	Offset amount	m	See Figure 155-1. See "External load influence diagram" and "Output Bearing Specs" of each series.

Load coefficient

Table 156-1

Load status	fw
During smooth operation without impact or vibration	1 to 1.2
During normal operation	1.2 to 1.5
During operation with impact or vibration	1.5 to 3

How to calculate the life during oscillating motion

HPGP HPG CSG-GH CSF-GH HPF

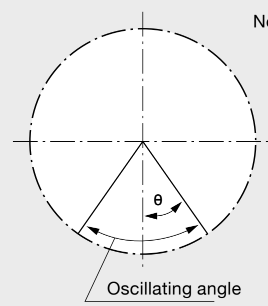
Calculate the life of the cross roller bearing during oscillating motion by Formula 156-3.

Formula 156-3

$$L_{oc}=\frac{10^6}{60\times n_1}\times\frac{90}{\theta}\times\left(\frac{C}{fw\cdot Pc}\right)^{10/3}$$

Loc	Rated life under oscillating motion	hour	-
n1	No. of reciprocating oscillation per min.	cpm	-
C	Basic dynamic load rating	N (kgf)	See "Output Bearing Specs."
Pc	Dynamic equivalent load	N (kgf)	See Formula 156-2.
fw	Load coefficient	—	See Table 156-1.
θ	Oscillating angle /2	Deg.	See Figure 156-1.

Figure 156-1



Note: When the oscillating angle is small (5° or less), it is difficult to generate an oil film on the contact surface of the orbit ring and the rolling element and fretting corrosion may develop.

Note When it is used for a long time while the rotation speed of the output shaft is in the ultra-low operation range (0.02rpm or less), the lubrication of the bearing becomes insufficient, resulting in deterioration of the bearing or increased load in the output side. When using it in the ultra-low operation range, contact us.

How to calculate the static safety coefficient

HPGP HPG CSG-GH CSF-GH HPF

In general, the basic static load rating (Co) is considered to be the permissible limit of the static equivalent load. However, obtain the limit based on the operating and required conditions. Calculate the static safety coefficient (fs) of the cross roller bearing using Formula 156-4.

General values under the operating condition are shown in Table 156-2. You can calculate the static equivalent load (Po) using Formula 156-5.

Formula 156-4

$$fs=\frac{Co}{Po}$$

Co	Basic static load	N (kgf)	See "Output Bearing Specs."
Po	Static equivalent load	N (kgf)	See Formula 156-5.

Formula 156-5

$$Po=Fr_{max}+\frac{2M_{max}}{dp}+0.44Fa_{max}$$

Fr_{max}	Max. radial load	N (kgf)	See "How to calculate the max. moment load."
Fa_{max}	Max. axial load	N (kgf)	
M_{max}	Max. moment load	Nm (kgfm)	
dp	Pitch Circle	m	See "Output Bearing Specs" of each series.

Static safety coefficient

Table 156-2

Load status	fs
When high precision is required	≧3
When impact or vibration is expected	≧2
Under normal operating condition	≧1.5

Input Bearing Specifications and Checking Procedure

Check the maximum load and life of the bearing on the input side if the reducer is an HPG input shaft unit or an HPF hollow shaft unit.

Checking procedure HPG HPF

(1) Checking maximum load

Calculate:
Maximum moment load (Mi *max*)
Maximum axial load (Fai *max*)
Maximum radial load (Fri *max*)

Maximum moment load (Mi *max*) ≦ Allowable moment load (Mc)
Maximum axial load (Fai *max*) ≦ Allowable axial load (Fac)
Maximum radial load (Fri *max*) ≦ Allowable radial load (Frc)

(2) Checking the life

Calculate:
Average moment load (Mi *av*)
Average axial load (Fai *av*)
Average input speed (Ni *av*)

Calculate the life and check it.

Specification of input bearing

Specification of input bearing HPG

Table 157-1

Size	Basic load rating			
	Basic dynamic load rating Cr		Basic static load rating Cor	
	N	kgf	N	kgf
11	2700	275	1270	129
14	5800	590	3150	320
20	9700	990	5600	570
32	22500	2300	14800	1510
50	35500	3600	25100	2560
65	51000	5200	39500	4050

Table 157-2

Size	Allowable moment load Mc		Allowable axial load Fac *1		Allowable radial load Frc *2	
	Nm	kgfm	N	kgf	N	kgf
11	0.16	0.016	245	25	20.6	2.1
14	6.3	0.64	657	67	500	51
20	13.5	1.38	1206	123	902	92
32	44.4	4.53	3285	335	1970	201
50	96.9	9.88	5540	565	3226	329
65	210	21.4	8600	878	5267	537

Specification of input shaft bearing HPF

Table 157-3

Size	Basic load rating			
	Basic dynamic load rating Cr		Basic static load rating Cor	
	N	kgf	N	kgf
25	14500	1480	10100	1030
32	29700	3030	20100	2050

Table 157-4

Size	Allowable moment load Mc		Allowable axial load Fac *1		Allowable radial load Frc *3	
	Nm	kgfm	N	kgf	N	kgf
25	10	1.02	1538	157	522	53.2
32	19	1.93	3263	333	966	98.5

〔Note：Table 157-2 and 157-4〕
*1 The allowable axial load is the value of an axial load applied along the axis of rotation.
*2 The allowable radial load of HPG series is the value of a radial load applied at the mid-point of the input shaft.
*3 The allowable radial load of HPG series is the value of a radial load applied to the point of 20 mm from the shaft edge (input flange edge).

Calculating maximum moment load ON input shaft

The maximum moment load (Mi_{max}) is calculated as follows.
Check that the following formulas are established in all circumstances:

Formula 158-1

$$Mi\ max = Fri\ max \cdot Lri + Fai\ max \cdot Lai$$

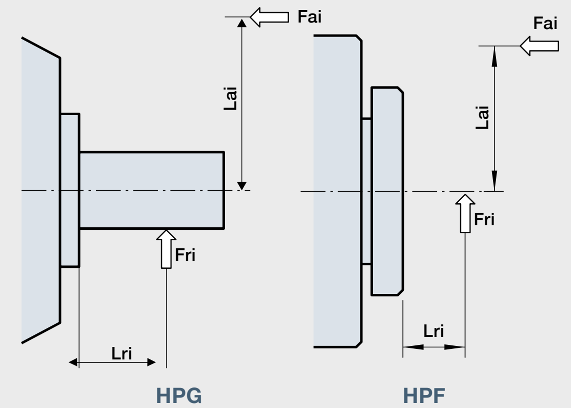
Fri *max*	Max. radial load	N (kgf)	See Fig. 158-1.
Fai *max*	Max. axial load	N (kgf)	See Fig. 158-1.
Lri,Lai	————	m	See Fig. 158-1.

$Mi\ max \leqq Mc$ (Allowable moment load)
$Fai\ max \leqq Fac$ (Allowable axial load)

HPG HPF

Figure 158-1

External load influence diagram

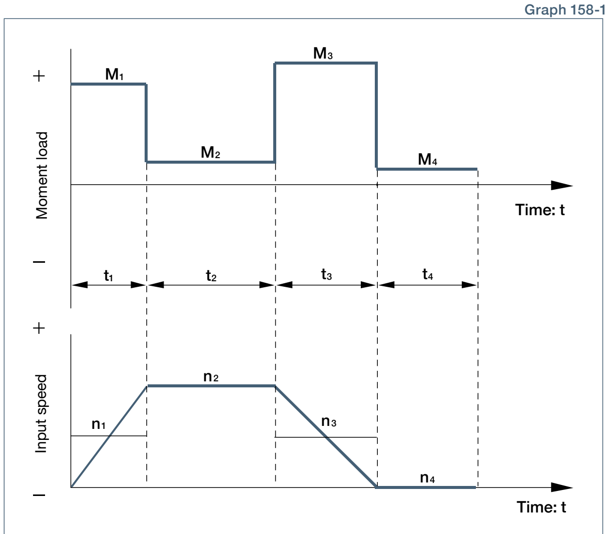


How to calculate average load
(Average moment load, average axial load, average input speed)

HPG HPF

If moment load and axial load fluctuate, they should be converted into the average load to check the life of the bearing.

Graph 158-1



How to calculate the average moment load (Mi_{av})

Formula 158-2

$$Mi\ av = \sqrt[3]{\frac{n_1t_1(|Mi_1|)^3 + n_2t_2(|Mi_2|)^3 \cdots n_nt_n(|Mi_n|)^3}{n_1t_1 + n_2t_2 + \cdots + n_nt_n}}$$

How to calculate the average axial load (Fai_{av})

Formula 158-3

$$Fai\ av = \sqrt[3]{\frac{n_1t_1(|Fai_1|)^3 + n_2t_2(|Fai_2|)^3 \cdots n_nt_n(|Fai_n|)^3}{n_1t_1 + n_2t_2 + \cdots + n_nt_n}}$$

How to calculate the average input speed (Ni_{av})

Formula 158-4

$$Ni\ av = \frac{n_1t_1 + n_2t_2 + \cdots + n_nt_n}{t_1 + t_2 + \cdots + t_n}$$

Calculating life of input bearing

Calculate the bearing life according to Calculation Formula 158-5 and check the life.

Formula 158-5

$$L_{10} = \frac{10^6}{60 \times Ni\ av} \times \left(\frac{Cr}{Pci}\right)^3$$

L_{10}	Life	Hour	—
Ni *av*	Average input speed	rpm	See Formula 158-4
Cr	Basic dynamic load rating	N (kgf)	See Table 157-1 and -3
Pci	Dynamic equivalent load	N	See Table 158-1 and -2

Dynamic equivalent load HPG

Table 158-1

Size	Pci
11	0.444 × Mi *av* + 1.426 × Fai *av*
14	0.137 × Mi *av* + 1.232 × Fai *av*
20	0.109 × Mi *av* + 1.232 × Fai *av*
32	0.071 × Mi *av* + 1.232 × Fai *av*
50	0.053 × Mi *av* + 1.232 × Fai *av*
65	0.041 × Mi *av* + 1.232 × Fai *av*

Dynamic equivalent load HPF

Table 158-2

Size	Pci
25	121 × Mi *av* + 2.7 × Fai *av*
32	106 × Mi *av* + 2.7 × Fai *av*

Mi*av*	Average moment load Nm (kgfm)	See Formula 158-2
Fai*av*	Average axial load N (kgf)	See Formula 158-3

Assembly

Assemble and mount your gearhead in accordance with these instructions to achieve the best performance.
Be sure to use the recommended bolts and use a torque wrench to achieve the proper tightening torques as recommended in tables below.

Motor assembly procedure HPGP HPG CSG-GH CSF-GH HPN

To properly mount the motor to the gearhead, follow the procedure outlined below, refer to figure 159-1

(1) Turn the input shaft coupling and align the bolt head with the rubber cap hole.

(2) With the speed reducer in an upright position as illustrated in the figure below, slowly insert the motor shaft into the coupling of speed reducer. Slide the motor shaft without letting it drop down. If the speed reducer cannot be positioned upright, slowly insert the motor shaft into the coupling of speed reducer, then tighten the motor bolts evenly until the motor flange and gearhead flange are in full contact. Exercise care to avoid tilting the motor when inserting it into the gear head.

(3) Tighten the input shaft coupling bolt to the recommended torque specified in the table below. The bolt(s) or screw(s) is (are) already inserted into the input coupling when delivered. Check the bolt size on the confirmation drawing provided.

Bolt tightening torque

Table 159-1

Bolt size		M3	M4	M5	M6	M8	M10	M12
Tightening torque	Nm	2.0	4.5	9.0	15.3	37.2	73.5	128
	kgfm	0.20	0.46	0.92	1.56	3.8	7.5	13.1

Caution: Always tighten the bolts to the tightening torque specified in the table above. If the bolt is not tightened to the torque value recommended slippage of the motor shaft in the shaft coupling may occur. The bolt size will vary depending on the size of the gear and the shaft diameter of the mounted motor. Check the bolt size on the confirmation drawing provided.

Two setscrews need to be tightened on size 11. See the outline dimensions on page 22 (HPGP) and page 34 (HPG standard) and page 46 (HPG helical). Tighten the screws to the tightening torque specified below.

Table 159-2

Bolt size		M3
Tightening torque	Nm	0.69
	kgfm	0.07

(4) Fasten the motor to the gearhead flange with bolts.

Bolt* tightening torque

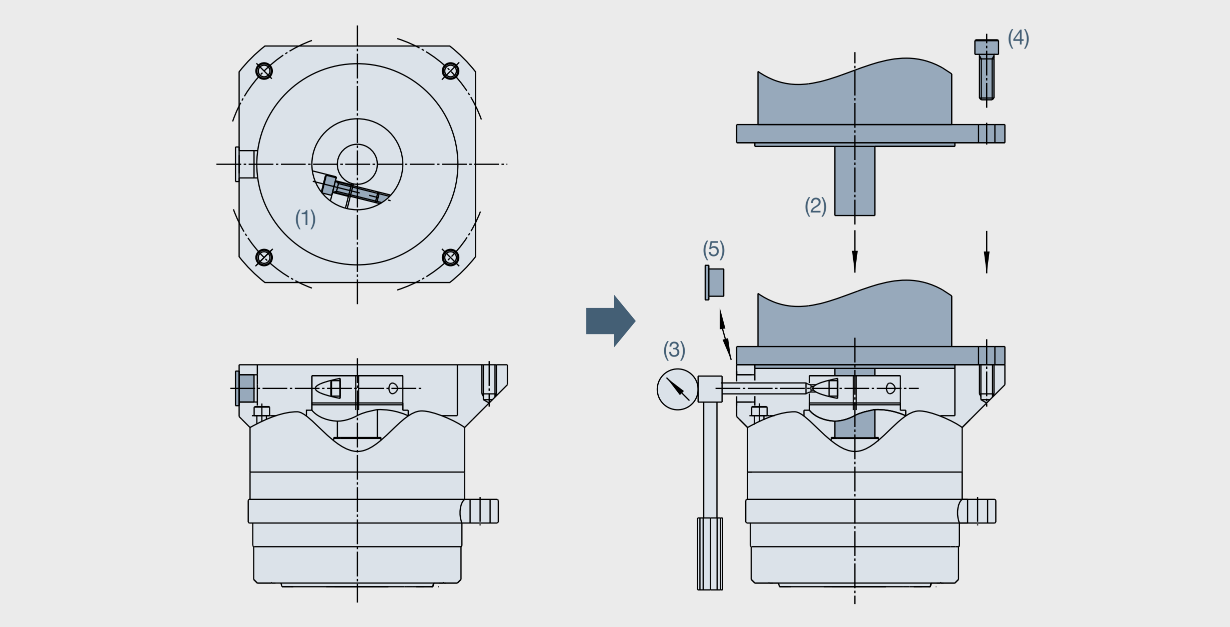
Table 159-3

Bolt size		M2.5	M3	M4	M5	M6	M8	M10	M12
Tightening torque	Nm	0.59	1.4	3.2	6.3	10.7	26.1	51.5	89.9
	kgfm	0.06	0.14	0.32	0.64	1.09	2.66	5.25	9.17

∗Recommended bolt: JIS B 1176 Hexagon socket head bolt, Strength: JIS B 1051 12.9 or higher
Caution: Be sure to tighten the bolts to the tightening torques specified in the table.

(5) Insert the rubber cap provided. This completes the assembly. (Size 11: Fasten screws with a gasket in two places)

Figure 159-1



Technical Information / Handling Explanation

Speed reducer assembly

HPGP HPG CSG-GH CSF-GH HPF HPN

Some right angle gearhead models weigh as much as 60 kg. No thread for an eyebolt is provided because the mounting orientation varies depending on the customer's needs. When mounting the reducer, hoist it using a sling paying extreme attention to safety.
When assembling gearheads into your equipment, check the flatness of your mounting surface and look for any burrs on tapped holes. Then fasten the flange (Part A in the diagram below) using appropriate bolts.

Bolt* tightening torque for flange (Part A in the diagram below)

Table 160-1

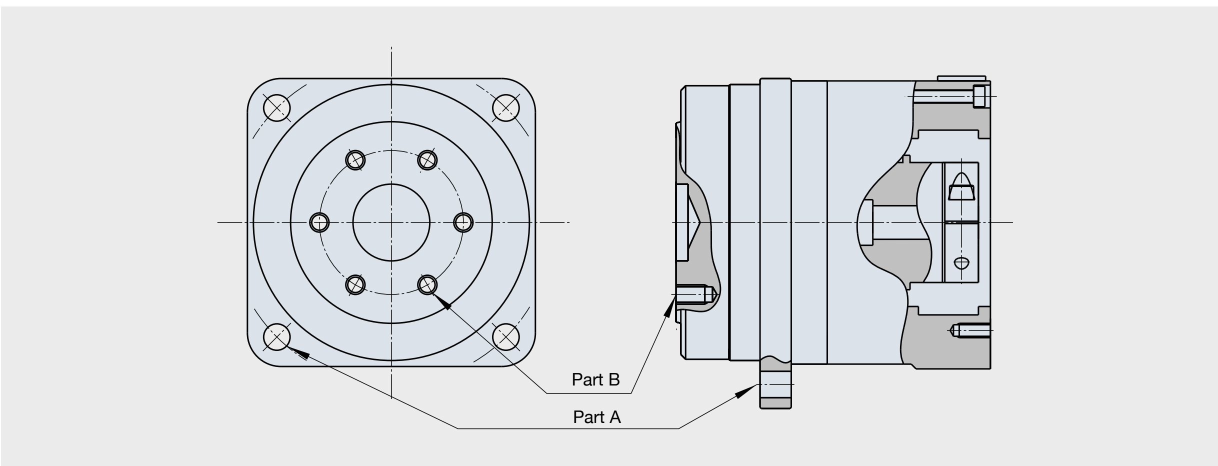
Size		HPN					HPGP / HPG / CSG-GH / CSF-GH						HPF	
		11	14	20	32	40	11	14	20	32	45/50	65	25	32
Number of bolts		4	4	4	4	4	4	4	4	4	4	4	12	12
Bolt size		M3	M5	M6	M8	M10	M3	M5	M8	M10	M12	M16	M4	M5
Mounting PCD	mm	50	70	100	130	165	46	70	105	135	190	260	127	157
Tightening torque	Nm	1.4	6.3	10.7	26.1	51.5	1.4	6.3	26.1	51.5	103	255	4.5	9.0
	kgfm	0.14	0.64	1.09	2.66	5.26	0.14	0.64	2.66	5.25	10.5	26.0	0.46	0.92
Transmission torque	Nm	27.9	110	223	528	1063	26.3	110	428	868	2030	5180	531	1060
	kgfm	2.85	11.3	22.8	53.9	108.5	2.69	11.3	43.6	88.6	207	528	54.2	108

* Recommended bolts: JIS B 1176 "Hexagon socket head bolts." Strength classification 12.9 or higher in JIS B 1051.

Mounting the load to the output flange

Follow the specifications in the table below when mounting the load onto the output flange.

Figure 160-1



Output flange mounting specifications

Bolt* tightening torque for output flange (Part B in the Figure 160-1) HPGP

Table 160-2

Size		11	14	20	32	50	65
Number of bolts		4	8	8	8	8	8
Bolt size		M4	M4	M6	M8	M12	M16
Mounting PCD	mm	18	30	45	60	90	120
Tightening torque	Nm	4.5	4.5	15.3	37.2	128.4	319
	kgfm	0.46	0.46	1.56	3.8	13.1	32.5
Transmission torque	Nm	25.3	84	286	697	2407	5972
	kgfm	2.58	8.6	29.2	71.2	245	609

* Recommended bolts: JIS B 1176 "Hexagon socket head bolts." Strength classification 12.9 or higher in JIS B 1051.

Bolt* tightening torque for output flange (Part B in the Figure 160-1) HPG

Table 160-3

Size		11	14	20	32	50	65
Number of bolts		3	6	6	6	14	6
Bolt size		M4	M4	M6	M8	M8	M16
Mounting PCD	mm	18	30	45	60	100	120
Tightening torque	Nm	4.5	4.5	15.3	37.2	37.2	319
	kgfm	0.46	0.46	1.56	3.8	3.80	32.5
Transmission torque	Nm	19.0	63	215	524	2036	4480
	kgfm	1.9	6.5	21.9	53.4	207.8	457

* Recommended bolts: JIS B 1176 "Hexagon socket head bolts." Strength classification 12.9 or higher in JIS B 1051.

Mounting the load to the output flange

Bolt* tightening torque for output flange (Part B in Figure 160-1) **CSG-GH**

Table 161-1

Size		14	20	32	45	65
Number of bolts		8	8	10	10	10
Bolt size		M4	M6	M8	M12	M16
Mounting PCD	mm	30	45	60	94	120
Tightening torque	Nm	4.5	15.3	37	128	319
	kgfm	0.46	1.56	3.8	3.1	32.5
Transmission torque	Nm	84	287	867	3067	7477
	kgfm	8.6	29.3	88.5	313	763

Bolt* tightening torque for output flange (Part B in Figure 160-1) **CSF-GH**

Table 161-2

Size		14	20	32	45	65
Number of bolts		6	6	6	16	8
Bolt size		M4	M6	M8	M8	M16
Mounting PCD	mm	30	45	60	100	120
Tightening torque	Nm	4.5	15.3	37.2	37.2	319
	kgfm	0.46	1.56	3.80	3.80	32.5
Transmission torque	Nm	63	215	524	2326	5981
	kgfm	6.5	21.9	53.4	237	610

Bolt* tightening torque for output flange (Part B in Figure 160-1) **HPF**

Table 161-3

Size		25	32
Number of bolts		12	12
Bolt size		M4	M5
Mounting PCD	mm	77	100
Tightening torque	Nm	4.5	9.0
	kgfm	0.46	0.92
Transmission torque	Nm	322	675
	kgfm	32.9	68.9

* Recommended bolts: JIS B 1176 "Hexagon socket head bolts." Strength classification 12.9 or higher in JIS B 1051.

Gearheads with an output shaft

HPN **HPG** **HPGP** **CSG-GH** **CSF-GH** **HPF**

Do not subject the output shaft to any impact when mounting a pulley, pinion or other parts.
An impact to the the output bearing may affect the speed reducer precision and may cause reduced life or failure.

Mechanical Tolerances

Superior mechanical precision is achieved by integrating the output flange with a high-precision cross roller bearing as a single component. The mechanical tolerances of the output shaft and mounting flange are specified below.

Figure 162-1

Output Flange: F0 (flange)

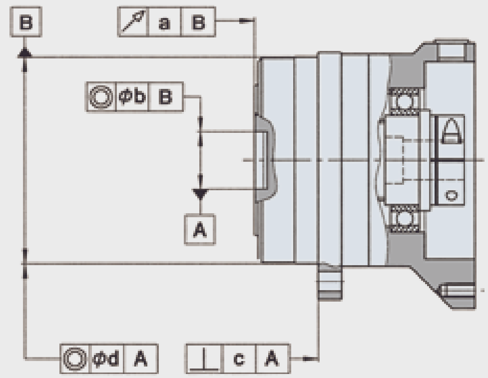
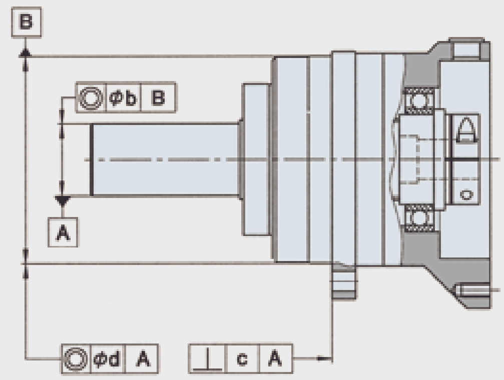


Figure 162-2

Output shaft: J2 [J20], J6 [J60] (shaft output)



HPGP HPG CSG-GH CSF-GH

Table 162-1

Size	Axial runout of output flange a	Radial runout of output flange pilot or output shaft b	Perpendicularity of mounting flange c	Concentricity of mounting flange d
11	0.020	0.030	0.050	0.040
14	0.020	0.040	0.060	0.050
20	0.020	0.040	0.060	0.050
32	0.020	0.040	0.060	0.050

HPGP HPG

Table 162-2

50	0.020	0.040	0.060	0.050
65	0.040	0.060	0.090	0.080

CSG-GH CSF-GH

Table 162-3

45	0.020	0.040	0.060	0.050
65	0.020	0.040	0.060	0.050

HPF

Table 162-4

25	0.020	0.040	0.060	0.050
32	0.020	0.040	0.060	0.050

* T.I.R.: Total indicator reading

(T.I.R.* Unit: mm)

Lubrication

Prevention of grease and oil leakage

(Common to all models)

- Only use the recommended greases.
- Provisions for proper sealing to prevent grease leakage are incorporated into the gearheads. However, please note that some leakage may occur depending on the application or operating condition. Discuss other sealing options with our applications engineers.
- When mounting the gearhead horizontally, position the gearhead so that the rubber cap in the adapter flange is facing upwards.

(CSG/CSF-GH Series)

- Contact us when using HarmonicDrive® CSG/CSF-GH series with the output shaft facing downward (motor on top) at a constant load or rotating continuously in one direction.

Sealing

(Common to all models)

- Provisions for proper sealing to prevent grease leakage from the input shaft are incorporated into the gearhead.
- A double lip Teflon oil seal is used for the output shaft (HPGP/HPG uses a single lip seal), gaskets or o-rings are used on all mating surfaces, and non contact shielded bearings are used for the motor shaft coupling (Double sealed bearings (D type) are available as an option*) . On the CSG/CSF-GH series, non contact shielded bearing and a Teflon oil seal with a spring is used.
- Material and surface: Gearbox: Aluminum, corrosion protected roller bearing steel, carbon steel (output shaft).
  Adapter flange: (if provided by Harmonic Drive) high-strength aluminum or carbon steel. Screws: black phosphate. The ambient environment should not subject any corrosive agents to the above mentioned material. The product provides protection class IP 54 under the provision that corrosion from the ambient atmosphere (condensation, liquids or gases) at the running surface of the output shaft seal is prevented. If necessary, the adapter flange can be sealed by means of a surface seal (e.g. Loctite 515).

* D type: Bearing with a rubber contact seal on both sides

(HPG/HPGP/HPF/HPN Series)

- Using the double sealed bearing (D type) for the HPGP/HPG series gearhead will result in a slightly lower efficiency compared to the standard product.
- An oil seal without a spring is used ON the input side of HPG series with an input shaft (HPG-1U) and HPF series hollow shaft reducer. An option for an oil seal with a spring is available for improved seal reliability, however, the efficiency will be slightly lower (available for HPF and HPG series for sizes 14 and larger).
- Do not remove the screw plug and seal cap of the HPG series right angle gearhead. Removing them may cause leakage of grease or affect the precision of the gear.

Standard Lubricants

HPG/HPGP/HPF/HPN Series

The standard lubrication for the HPG/HPGP/HPF/HPN series gearheads is grease.
All gearheads are lubricated at the factory prior to shipment and additional application of grease during assembly is not required.
The gearheads are lubricated for the life of the gear and do not require re-lubrication.
High efficiency is achieved through the unique planetary gear design and grease selection.

Lubricants

Harmonic Grease SK-2 (HPGP/HPG-14, 20, 32)
Manufacturer: Harmonic Drive Systems Inc.

Base oil: Refined mineral oil	Consistency: 265 to 295 at 25°C
Thickening agent: Lithium soap	Dropping point: 198°C
Additive: Extreme pressure agent and other	Color: Green
Standard: NLGI No. 2	

EPNOC Grease AP (N) 2 (HPGP/HPG-11, 50, 65／HPF-25, 32)
Manufacturer: Nippon Oil Co.

Base oil: Refined mineral oil	Consistency: 282 at 25°C
Thickening agent: Lithium soap	Dropping point: 200°C
Additive: Extreme pressure agent and other	Color: Light brown
Standard: NLGI No. 2	

PYRONOC UNIVERSAL 00 (HPG right angle gearhead/HPN)
Manufacturer: Nippon Oil Co.

Base oil: Refined mineral oil	Consistency: 420 at 25°C
Thickening agent: Urea	Dropping point: 250°C or higher
Standard: NLGI No. 00	Color: Light yellow

MULTEMP AC-P (HPG-X-R)
Manufacturer: KYODO YUSHI CO, LTD

Base oil: Composite hydrocarbon oil and diester	Standard: NLGI No. 2
Thickening agent: Lithium soap	Consistency: 280 at 25°C
Additive: Extreme pressure and others	Dropping point: 200°C
	Color: Black viscose

Ambient operating temperature range: –10°C to +40°C

The lubricant may deteriorate if the ambient operating temperature is outside of recommended operating range. Please contact our sales office or distributor for operation outside of the ambient operating temperature range.
The temperature rise of the gear depends upon the operating cycle, ambient temperature and heat conduction and radiation based on the customers installation of the gear. A housing surface temperature of 70ºC is the maximum allowable limit.

CSG-GH/CSF-GH Series

The standard lubrication for the CGS-GH / CSF-GH series gearheads is grease.
All gearheads are lubricated at the factory prior to shipment and additional application of grease during assembly is not necessary.

Lubricants

Harmonic Grease SK-1A (Size 20, 32, 45, 65)
Manufacturer: Harmonic Drive Systems Inc.

This grease has been developed exclusively for HarmonicDrive® gears and is excellent in durability and efficiency compared to commercial general-purpose grease.

Base oil: Refined mineral oil
Thickening Agent: Lithium soap
Additive: Extreme pressure agent and other
Standard: NLGI No. 2
Consistency: 265 to 295 at 25°C
Dropping point: 197°C
Color: Yellow

Harmonic Grease SK-2 (Size 14)
Manufacturer: Harmonic Drive Systems Inc.

This grease has been developed exclusively for smaller sized HarmonicDrive® gears and allows smooth wave generator rotation.

Base oil: Refined mineral oil
Thickening Agent: Lithium soap
Additive: Extreme pressure agent and other
Standard: NLGI No. 2
Consistency: 265 to 295 at 25°C
Dropping point: 198°C
Color: Green

Ambient operating temperature range: –10°C to +40°C

The lubricant may deteriorate if the ambient operating temperature is outside the recommended temperature range. Please contact our sales office or distributor for operation outside of the ambient operating temperature range.
The temperature rise of the gear depends upon the operating cycle, ambient temperature and heat conduction and radiation based on the customers installation of the gear. A housing surface temperature of 70°C is the maximum allowable limit.

When to change the grease

The life of the Harmonic Drive® gear is affected by the grease performance. The grease performance varies with temperature and deteriorates at elevated temperatures. Therefore, the grease will need to be changed sooner than usual when operating at higher temperatures. The graph on the right indicates when to change the grease based upon the temperature (when the average load torque is less than or equal to the rated output torque at 2000 rpm). Also, using the formula below, you can calculate when to change the grease when the average load torque exceeds the rated output torque (at 2000 rpm).

Formula to calculate the grease change interval when the average load torque exceeds the rated torque

Formula 164-1

$$L_{GT} = L_{GTn} \times \left(\frac{Tr}{Tav}\right)^{3}$$

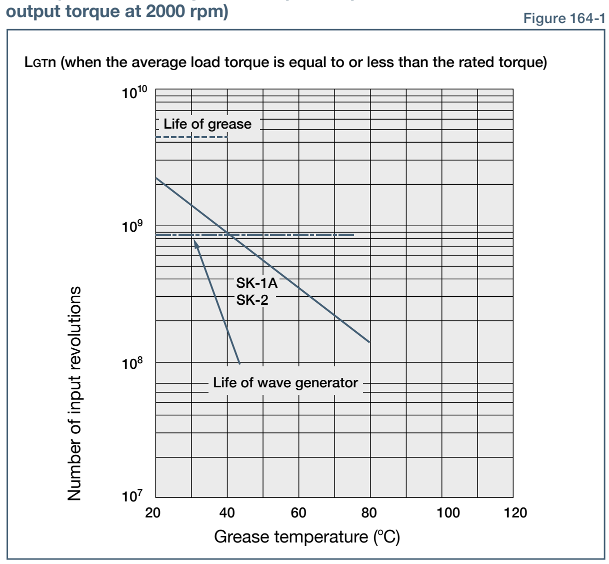
Formula symbols

Table 164-1

L_{GT}	Grease change interval when Tav > Tr	Input rotations	————
L_{GTn}	Grease change interval when Tav <= Tr	Input rotations	See Graph 164-1
Tr	Output torque at 2000 rpm	Nm, kgfm	See the "Rating table" on pages 87 & 97.
T_{av}	Average load torque	Nm, kgfm	Calculation formula: See page 111.

When to change the grease:
LGTn (when the average load torque is equal to or less than the rated output torque at 2000 rpm)

Figure 164-1



* L10 Life of wave generator bearing

Reference values for grease refill amount

Table 164-2

Size	14	20	32	45	65
Amount: g	0.8	3.2	6.6	11.6	78.6

Precautions when changing the grease

Strictly observe the following instructions when changing the grease to avoid problems such as grease leakage or increase in running torque.

- **Note that the amount of grease listed in Table 164-2 is the amount used to lubricate the gear at assembly. This should be used as a reference. Do not exceed this amount when re-greasing the gearhead.**
- **Remove grease from the gearhead and refill it with the same quantity. The adverse effects listed above normally do not occur until the gear has been re-greased 2 times. When re-greasing 3 times or more, it is essential to remove grease (using air pressure or other means) before re-lubricating with the same amount of grease that was removed.**

Warranty

Please contact us or visit our website at www.harmonicdrive.net for warranty details for your specific product.

All efforts have been made to ensure that the information in this catalog is complete and accurate. However, Harmonic Drive LLC is not liable for any errors, omissions or inaccuracies in the reported data. Harmonic Drive LLC reserves the right to change the product specifications, for any reason, without prior notice. For complete details please refer to our current Terms and Conditions posted on our website.

Disposal

When disposing of the product, disassemble it and sort the component parts by material type and dispose of the parts as industrial waste in accordance with the applicable laws and regulations. The component part materials can be classified into three categories.

(1) Rubber parts: Oil seals, seal packings, rubber caps, seals of shielded bearings on input side (D type only)
(2) Aluminum parts: Housings, motor flanges
(3) Steel parts: Other parts

Trademark

HarmonicDrive® is a registered trademark of Harmonic Drive LLC.
HarmonicPlanetary® is a registered trademark of Harmonic Drive LLC.

Safety

⚠ **Warning** : Means that improper use or handling could result in a risk of death or serious injury.

⚠ **Caution** : Means that improper use or handling could result in personal injury or damage to property.

Application Restrictions

This product cannot be used for the following applications:

- * Space flight hardware
- * Vacuum environments
- * Equipment for transport of humans
- * Aircraft equipment
- * Automotive equipment
- * Nuclear power equipment
- * Personal recreation equipment
- * Equipment for use in a special environment
- * Equipment and apparatus used in residential dwellings
- * Equipment that directly works on human bodies
- * Medical equipment

Please consult Harmonic Drive LLC beforehand if intending to use one of our product for the aforementioned applications.
Fail-safe devices that prevent an accident must be designed into the equipment when the products are used in any equipment that could result in personal injury or damage to property in the event of product failure.

Design Precaution: Be certain to read the catalog when designing the equipment.

Caution

Use only in the proper environment.
- Please ensure to comply with the following environmental conditions:
  - Ambient temperature 0 to 40℃
  - No splashing of water or oil
  - Do not expose to corrosive or explosive gas
  - No dust such as metal powder

Caution

Install the equipment properly.
- Carry out the assembly and installation precisely as specified in the catalog.
- Observe our recommended fastening methods (including bolts used and tightening torques).
- Operating the equipment without precise assembly can cause problems such as vibration, reduction in life, deterioration of precision and product failure.

Caution

Install the equipment with the required precision.
- Design and assemble parts to keep all catalog recommended tolerances for installation.
- Failure to hold the recommended tolerances can cause problems such as vibration, reduction in life, deterioration of precision and product failure.

Caution

Use the specified lubricant.
- Using other than our recommended lubricant can reduce the life of the product. Replace the lubricant as recommended.
- Gearheads are factory lubricated. Do not mix installed lubricant with other kinds of grease.

Operational Precaution: Be certain to read the catalog before operating the equipment.

Caution

Use caution when handling the product and parts.
- Do not hit the gear or any part with a hammer.
- If you use the equipment in a damaged condition, the gearhead may not perform to catalog specifications. It can also cause problems including product failure.

Caution

Operate within the allowable torque range.
- Do not apply torque exceeding the momentary peak torque. Applying excess torque can cause problems such as loosened bolts, generation of backlash and product failure.
- An arm attached directly to the output shaft that strikes a solid object can damage the arm or cause the output of the gearhead to fail.

Caution

Do not alter or disassemble the product or parts.
- Harmonic Planetary® and Harmonic Drive® products are manufactured as matched sets. Catalog ratings may not be achieved if the component parts are interchanged.

Caution

Do not disassemble the products.
- Do not disassemble and reassemble the products. Original performance may not be achieved.

Warning

Do not use your finger to turn the gear.
- Do not insert your finger into the gear under any circumstances. The finger may get caught in the gear causing an injury.

Warning

Large sizes (45, 50 and 65) are heavy. Use caution when handling.
- They are heavy and may cause a lower-back injury or an injury if dropped on a hand or foot. Wear protective shoes and back support when handling the product.

Caution

Stop operating the system if any abnormality occurs.
- Shut down the system promptly if any abnormal sound or vibration is detected, the rotation has stopped, an abnormally high temperature is generated, an abnormal motor current value is observed or any other anomalies are detected. Continuing to operate the system may adversely affect the product or equipment.
- Please contact our sales office or distributor if any anomaly is detected.

Caution

- Rust-proofing was applied before shipping. However, please note that rusting may occur depending on the customers' storage environment.
- Although black oxide finish is applied to some of our products, it does not guarantee that rust will not form.

Handling Lubricant

Warning

Precautions on handling lubricants
- Lubricant in the eye can cause inflammation. Wear protective glasses to prevent it from getting in your eye.
- Lubricant coming in contact with the skin can cause inflammation. Wear protective gloves when you handle the lubricant to prevent it from contacting your skin.
- Do not ingest (to avoid diarrhea and vomiting).
- Use caution when opening the container. There may be sharp edges that can cut your hand. Wear protective gloves.
- Keep lubricant out of reach of children.

Warning

First-aid
- Inhalation: Remove exposed person to fresh air if adverse effects are observed.
- Ingestion: Seek immediate medical attention and do not induce vomiting unless directed by medical personnel.
- Eyes: Flush immediately with water for at least 15 minutes. Get immediate medical attention.
- Skin: Wash with soap and water. Get medical attention if irritation develops.

Caution

Disposal of waste oil and containers
- Follow all applicable laws regarding waste disposal. Contact your distributor if you are unsure how to properly dispose of the material.
- Do not apply pressure to an empty container. The container may explode.
- Do not weld, heat, drill or cut the container. This may cause residual oil to ignite or cause an explosion.

Caution

Storage
- Tightly seal the container after use. Store in a cool, dry, dark place. Keep away from open flames and high temperatures.

Disposal

Caution

Please dispose of as industrial waste.
- Please dispose of the products as industrial waste when their useful life is over.

Major Applications of Our Products


Metal Working Machines


Processing Machine Tools


Measurement, Analytical and Test Systems


Medical Equipment


Telescopes

Source: National observatory of Inter-University Research Institute Corporation


Energy

Courtesy of Haliiburton/Sperry Drilling Services


Crating and Packaging Machines


Communication Equipment


Space Flight Hardware

Rover image created by Dan Maas, copyrighted to Cornell and provided courtesy NASA/ JPL-Caltech.

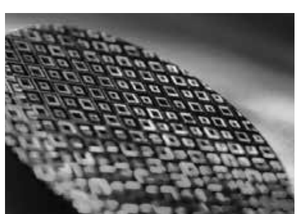

Glass and Ceramic Manufacturing Systems


Robots


Humanoid Robots

Source: Honda Motor Co., Ltd.


Printing, Bookbinding and Paper Machines

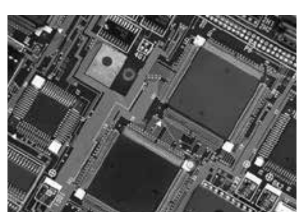

Semiconductor Manufacturing Equip.


Optical Equipment


Machine Tools

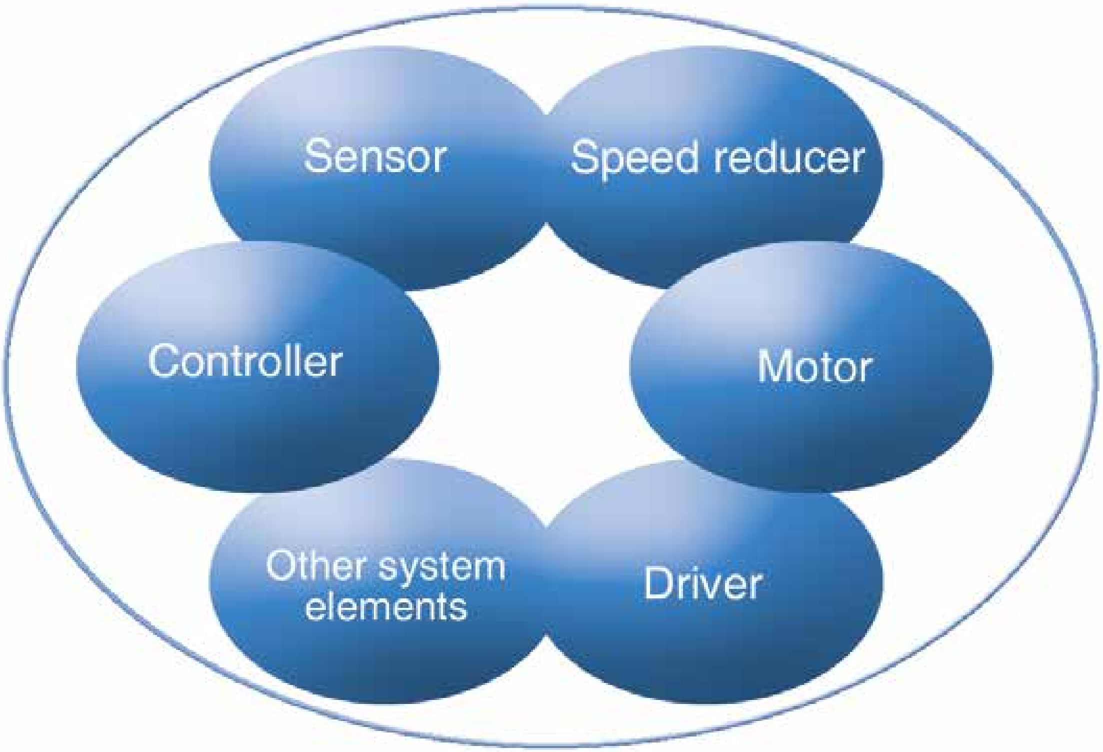

Paper-making Machines


Flat Panel Display Manufacturing Equip.


Printed Circuit Board Manufacturing Machines


Aerospace

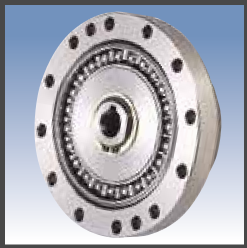
Experts in Precision Motion Control



Other Products

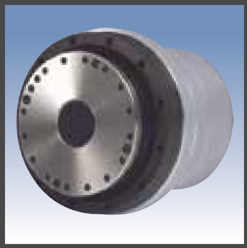
HarmonicDrive® Gearing

HarmonicDrive® speed reducer delivers precise motion control by utilizing the strain wave gearing principle.



Rotary Actuators

High-torque actuators combine performance matched servomotors with HarmonicDrive® gears to deliver excellent dynamic control characteristics.



Linear Actuators

Compact linear actuators combine a precision lead screw and HarmonicDrive® gear. Our versatile actuators deliver both ultra precise positioning and high torque.

CSF Mini Gearheads

CSF mini gearheads provide high positioning accuracy in a super-compact package.

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